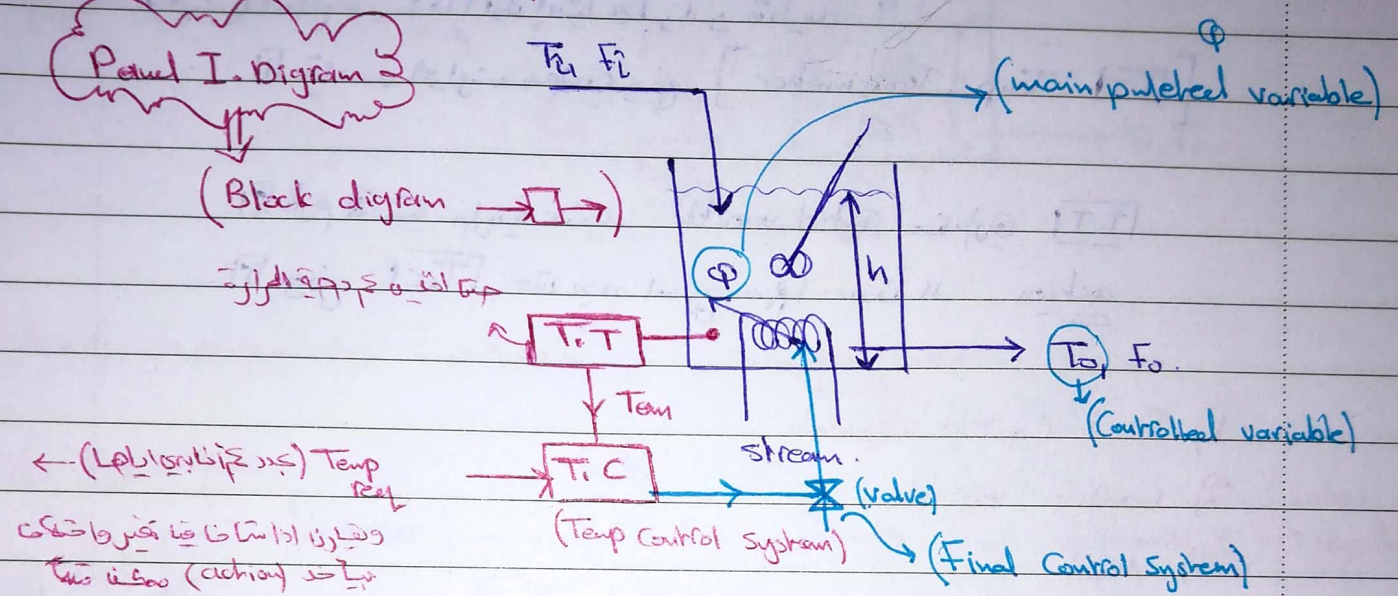
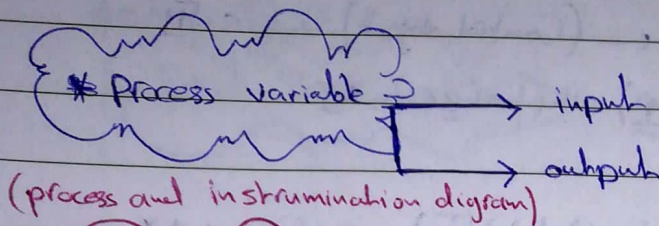




CONTROL

Dr. Deeb Abu-Fara

By: Sana'a Iqtait



* process variable $\Rightarrow T_i, F_i, T_o, F_o, h, \phi$

input $\Rightarrow T_i, F_i, \phi$

output $\Rightarrow F_o, T_o, h, \phi$

* a STH where the input is at inlet Temp T_i and Flow rate F_i and we give ϕ, T_o, F_o, T_i, F_i

* purpose of the process \Rightarrow Convey heat to change T_i to T_o

* My Control object \Rightarrow is the output Temp (T_o) and output Flow rate

(Temp sensor) (Valve) \Rightarrow stream \Rightarrow Flow \Rightarrow heat

heat \Rightarrow Flow \Rightarrow ϕ \Rightarrow Flow \Rightarrow heat

(Resistance) \Rightarrow Valve \Rightarrow stream \Rightarrow electrical heater \Rightarrow heat

element \Rightarrow Control system \Rightarrow (Final Control) element \Rightarrow stream \Rightarrow main valve \Rightarrow heat

Control object (Control object) ← output
 (Control variable) ← Temp

* Control object (To) → Control object

[T.T. Temp. Transducer] →

[T.T.] Control variable

[T.C.] → action

1) Final Control element

2) Control variable

3) Input

4) Control

5) ~~Measuring device~~ Measuring device

input

output

(main pulatced)

(Controlled)

To

main

input

(main pulatced) → (Controlled) → (mathematical relation ship)

dynamic process

dynamic model

(Unsteady state)

out & in → System → Control

dynamic model → (mathematical) reaction

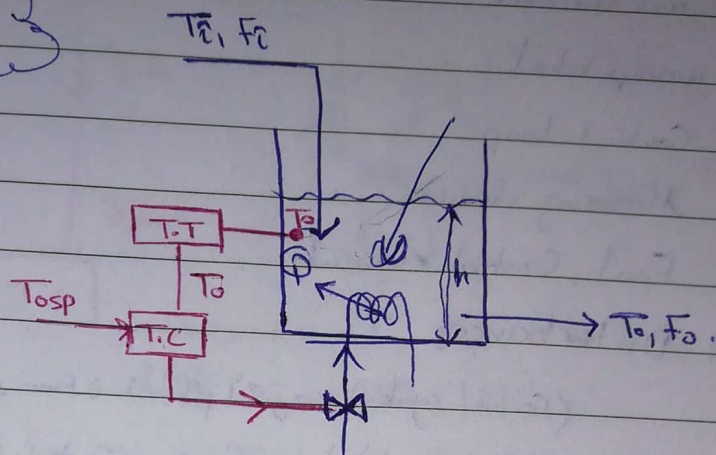
N O T E B O O K

(main pulatced variable) → Control

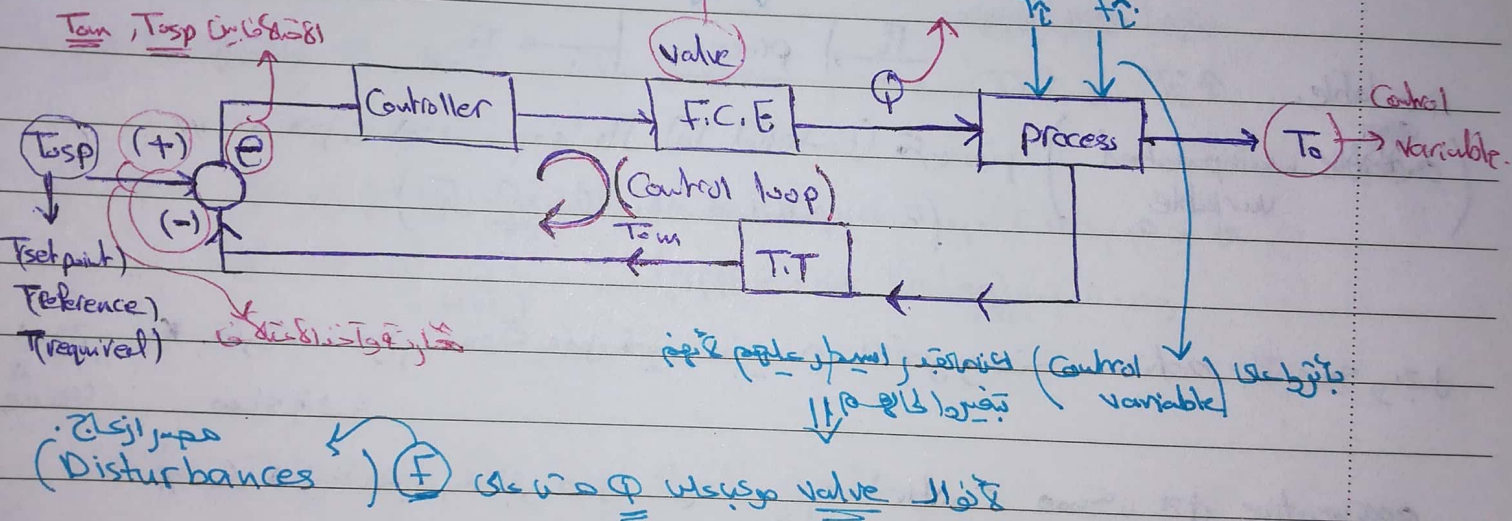
strategy II

Feed Back control

(جوابی کنترول)



Ques 1: In a feed-back control system, the manipulated variable is the valve.



Control variable is the manipulated variable (Control loop). The process is a function of the manipulated variable only. The process is a function of the manipulated variable only. The process is a function of the manipulated variable only.

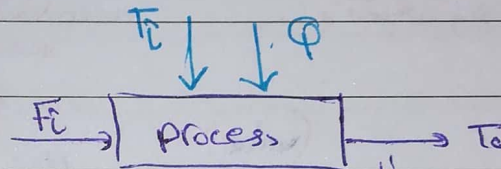
The process is a function of the manipulated variable only. The process is a function of the manipulated variable only. The process is a function of the manipulated variable only.

The process is a function of the manipulated variable only. The process is a function of the manipulated variable only. The process is a function of the manipulated variable only.

- 1) Control variable
- 2) manipulated
- 3) Control loop
- 4) Measuring device.
- 5) Final, Control elements
- 6) Disturbances

(Control system) (B.D) (Control system) (B.D)

Fe (valve) (manipulated variable) (Controlled To) (To) (Controlled To) (To)



available. (manipulated variable) (Controlled To) (To) (Controlled To) (To) (manipulated to Φ $\frac{dP}{dt}$ Fe)

action (Controlled To) (To) (Controlled To) (To) (Controlled To) (To) (Controlled To) (To)

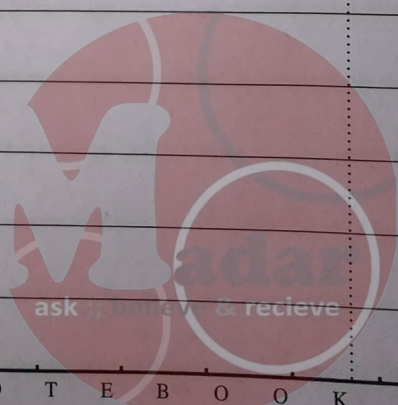
apprentive action (Controlled To) (To) (Controlled To) (To) (Controlled To) (To) (Controlled To) (To)

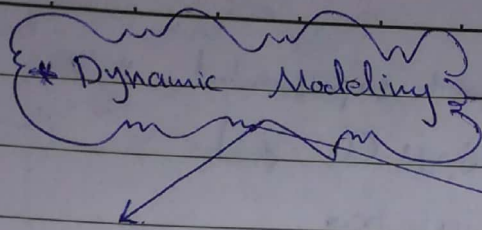
(input) (To) (Controlled To) (To) (Controlled To) (To) (Controlled To) (To)

apprentive \rightarrow (prevent) * \rightarrow T_o \rightarrow T_i
 This loop preventive to tracing change in T_o
 due to change in T_i only

\Downarrow
 deficiency \rightarrow T_o \rightarrow T_i
 (input change) \rightarrow (controlled) \rightarrow T_o
 on system loop
 \downarrow
 variable \rightarrow T_o

Feed Back * \rightarrow T_o \rightarrow T_i
 (Feed Back) \rightarrow T_o
 (deficiency) \rightarrow T_o
 \rightarrow T_i





→ in Unsteady state
change.

Theoretical (Analytical)

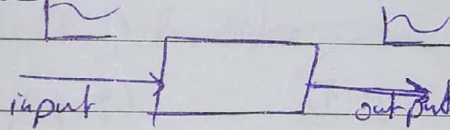
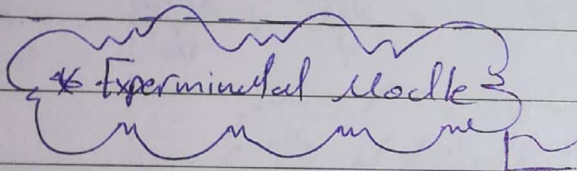
Experimental (Theoretical vs experimental comparison)

Basic equation (energy, mass, Momentum Balance)

Conservation of mass and energy.

Rate of change of mass and energy

Basic equation using Physical Rate equation



(input → output)

input will change → output will change

input is given, output is given (Deterministic ~~Model~~)

(Deterministic Model) & t

input is given, output is given (Stochastic Model) & t

(Stochastic Model) → input is given, output is given (Stochastic Model)

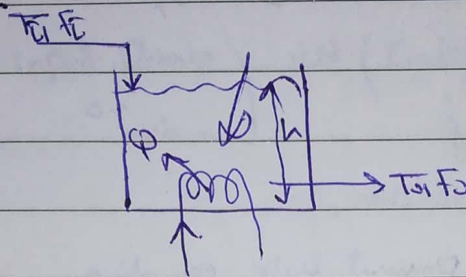
Analytical is easier than experimental

general is better than specific

Rate, Condition in Case of Process

40°C in 25°C

non linear, is a chemical process



* Mass Balance (instantaneous)

$$\frac{dm}{dt} = m_{in} - m_{out}$$

$$\frac{d(\rho_0 A h)}{dt} = \rho_1 F_1 - \rho_0 F_0 \quad (\rho_0 \neq \rho_1)$$

A = cross sectional Area.

let ($\rho = \text{constant}$)

constant $\rightarrow A \frac{dh}{dt} = F_1 - F_0$

physical, not mathematical, is better than mathematical

$$\frac{h}{R} \propto F_0$$

variable $\rightarrow (F_1, F_2)$

$$A \frac{dh}{dt} = F_1 - \frac{h}{R} F_0$$

constant $\rightarrow (A, R)$

R = Resistance to Flow.

(is a pipe or pipe or flow)

~~$$A \frac{dh}{dt} = F_1 - \frac{h}{R} F_0$$~~

$$AR \frac{dh}{dt} + h = R F_1$$

Mathematical is better than physical

(h) Function in

input (F_1) only

* is a physical or theoretical

$AR \frac{dh}{dt} + h = R F_i$ → Change in output h depends on change in input F_i only.

~~تغییر خروجی h به تغییر ورودی F_i بستگی دارد~~

* Energy Balance (Enthalpy Balance) &

$$\frac{d}{dt} [\underbrace{PAh C_p (T_o - T_{ref})}_{(mcp \Delta T)}] = [\underbrace{F_i C_p (T_i - T_{ref})}_{\text{enthalpy in}} - \underbrace{F_o C_p (T_o - T_{ref})}_{\text{enthalpy out}} + Q]$$

$T_{ref} = T_{env}$, $A_p C_p = \text{constant}$

$$\cancel{A_p C_p} \frac{d(h T_o)}{dt} = \frac{P C_p}{A} (F_i T_i) - \frac{P C_p}{A} (F_o T_o) + \frac{Q}{A_p C_p}$$

$$\frac{h dT_o}{dt} + \frac{T_o dh}{h} = \frac{1}{A} F_i T_i - \frac{1}{A} F_o T_o + \frac{Q}{A_p C_p}$$

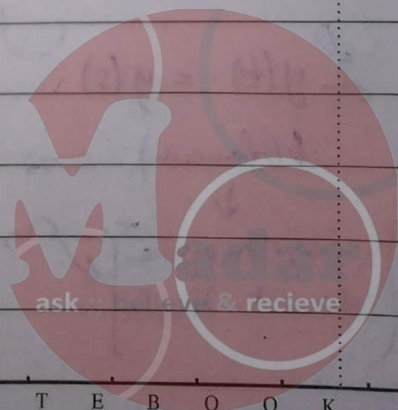
$\frac{dh}{dt} = \frac{R F_i - h}{RA}$ → ~~تغییر h به تغییر F_i بستگی دارد~~

~~تغییر h به تغییر F_i بستگی دارد~~

$$?? \frac{dT_o}{dt} + ?? T_o = ?? F_i T_i + ?? Q$$

~~تغییر T_o به تغییر F_i بستگی دارد~~

تغییر T_o به تغییر F_i بستگی دارد



$AR \frac{dh}{dt} + h = R F_j$ → variable steps
 Steady state = zero
 $0 + h_{ss} = R F_{jss}$ → $(h - h_{ss})$
 $AR \frac{dh}{dt} + (h - h_{ss}) = R (F_j - F_{jss})$

$$h - h_{ss} = h'$$

$$F_j - F_{jss} = F'$$

(Variable in the eqn) → Deviation Form

Deviation Form

Relationship between (Block diagram) B.D. and Transfer Function

Transfer Function is derived from B.D. by Laplace transform

Deviation Form

deviation form Laplace is Transfer Function

Laplace Transform → Algebraic equation
 differential equation → linear differential equation
 Deviation form. Laplace

$$\mathcal{L} y(t) = y(s)$$

$$t(\text{domain}) \rightarrow s(\text{domain})$$

domain

\downarrow
 Laplace Transform
 domain t → domain s

$$\rightarrow \mathcal{L}y(t) = y(s)$$

$$\mathcal{L}\left(\frac{dy}{dt}\right) = Sy(s) - y(0)$$

initial value of the function:

deviation $\epsilon = T_{me} - T_{true}$
موتغير و با اینه تغییر = خطا

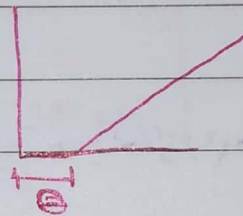
$$\rightarrow \mathcal{L} \frac{d^2 y}{dt^2} = s^2 y(s) - s y'(0) - y(0)$$

$$\rightarrow \mathcal{L} \int y(t) dt = \frac{1}{s} y(s)$$

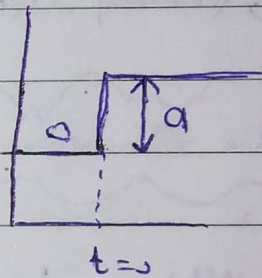
هذا انت على نوعي
Integral

$$\rightarrow \mathcal{L} y(t-\Theta) = e^{-\Theta s} y(s)$$

↓
delay time

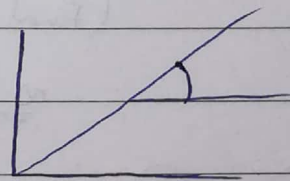


→ L step function = $\frac{a}{s}$



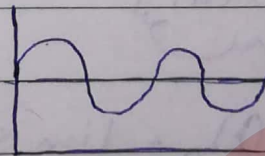
$$y(t) = 0 \quad t < 0$$

→ Ramp Function $y(t) = at$
 $y(s) = \frac{a}{s^2}$



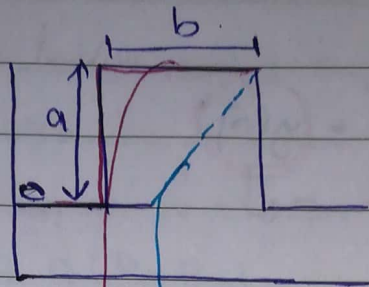
Sin wave Function.

$$y(s) = \frac{??}{s^2 + \dots}$$



→ Pulse Function.

$$y(s) = \frac{a(1 - e^{-bs})}{s}$$



is a step function
at t=0

$$\frac{a}{s} + \frac{-a(e^{-bs})}{s}$$

2 step in the
upside

$$\frac{a(1 - e^{-bs})}{s}$$

* → 2 step → one in positive

one in negative and the second
on delay function

(t) is a step function (Heaviside) which is a unit domain $\leftarrow (s) \times$

* Final value Theorem

$$y(t)_{\infty} = \lim_{s \rightarrow 0} s y(s)$$

$s \rightarrow 0$

$y(t)_{\infty}$ is a step function (Heaviside) is inverse Laplace of the algebraic series *

(Final value Theorem) is

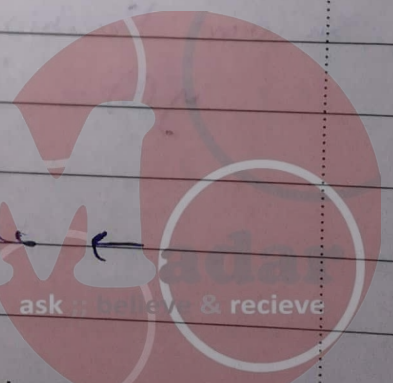
∞ is $y(t)$ is the inverse Laplace of the *

* Initial value Theorem

$$y(t)_0 = \lim_{s \rightarrow \infty} s y(s)$$

$s \rightarrow \infty$

→ (Transfer form) is a function



$$\mathcal{L} \left[AR \frac{dh'}{dt} + h' \right] = R F_i'$$

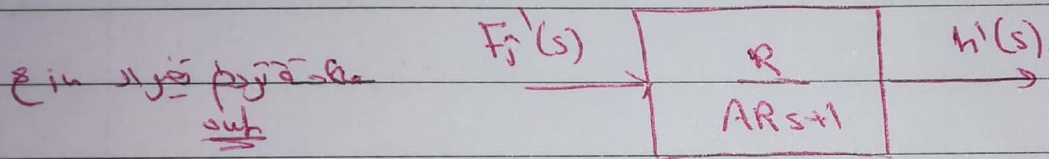
Laplace \mathcal{L} -T

$$AR [sh'(s) - \cancel{h'(0)}] + h'(s) = R F_i'$$

$$ARh'(s) [s+1] = R F_i'(s)$$

$$\frac{W(s)}{F_i'(s)} = \frac{h'(s)}{F_i'(s)} = \frac{R}{AR[s+1]} \Rightarrow \text{Transfer Function}$$

[change output \rightarrow change in ~~input~~]



Transfer Function \Rightarrow is the laplace transform of the ~~in~~ change in input to laplace transform of the change in output



* First order System

what is a first order system??

it is a system described by 1st order D.E

$$a_1 \frac{dy(t)}{dt} + a_0 y(t) = b x(t)$$

should reach the transfer function using Laplace.

$$\frac{a_0}{a_1} \frac{dy(t)}{dt} + y(t) = \frac{b}{a_1} x(t)$$

$$\frac{a_0}{a_1} [sY(s) - y(0)] + Y(s) = \frac{b}{a_1} X(s)$$

$$\frac{a_0}{a_1} \int Y(s) + Y(s) = \frac{b}{a_1} X(s) \Rightarrow Y(s) \left[\frac{a_0}{a_1} s + 1 \right] = \frac{b}{a_1} X(s)$$

$$G(s) = \frac{Y(s)}{X(s)} = \left[\frac{b/a_1}{\frac{a_0}{a_1} s + 1} \right] \rightarrow \text{Transfer Function}$$

(b/a₁)

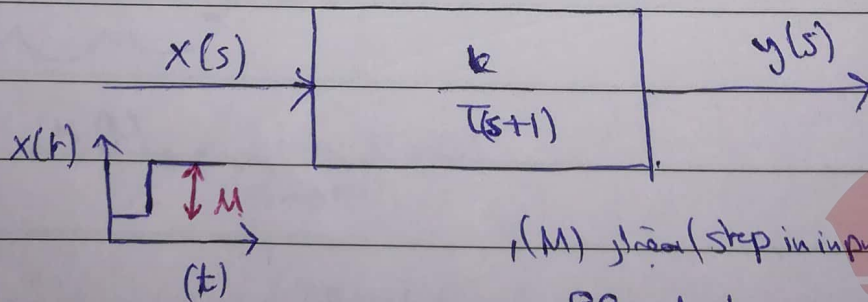
$$\rightarrow G(s) = \frac{k}{Ts + 1}$$

k: steady state Gain.

T: time Constant.

For any system k and T should be identified.

(k & T) are the parameters of the system.



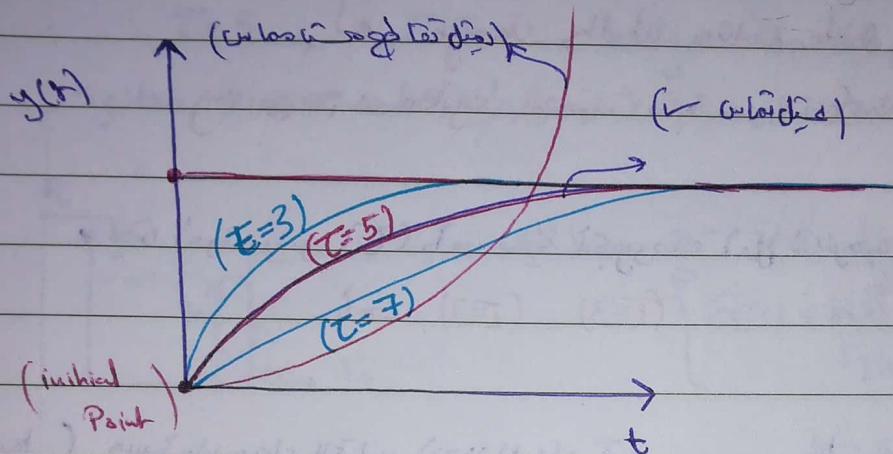
(M) step in input

output

$$y(s) = \frac{k}{Ts+1} * (X(s)) = \frac{M}{s}$$

$$\text{Lstep } \underline{g} = \frac{g}{s} \Rightarrow y(t) = \mathcal{L}^{-1} y(s)$$

$$y(t) = \mathcal{L}^{-1} \frac{k}{s} \xrightarrow{\text{مكافئته بـ } \frac{1}{s}} y(t) = kM(1 - e^{-t/T})$$



* لولم أيا سة يلو حة ؟

① قة البارة (t=0)

y(t) change → initial value of the change y(t) = zero → (y(t) = zero)

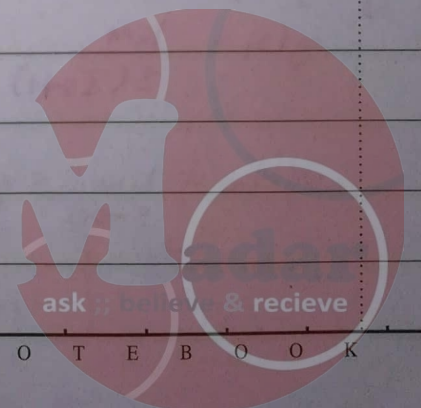
$$y(t) = kM \leftarrow \left(e^{-\infty/t} = \text{zero} \right) \leftarrow (t = \infty) \text{ قة البارة ②}$$

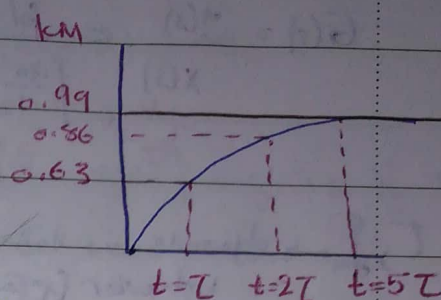
intensity ↓ إلى kM انوال Function ليو

والبارة انوال (t) قارب، انوال Function ليو

وراقب

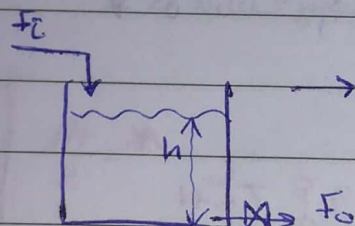
$$y(t) = kM \rightarrow \text{مادة مستقر}$$



$$G(s) = \frac{1}{s+1}$$


$T=5 \rightarrow T=5$ in steady state w.r.t. ω_{in}

System is parameterized by $\omega_{in} = \omega_{out} = \omega_{ref} = k_c T$



→ mass Balance @ AR $\frac{dh}{dt} + h = R F_p$

$$\left\{ \begin{aligned} G(s) &= \frac{Ks^{0.5}}{F_D(s)} = \frac{K}{ARs+1} \end{aligned} \right\} \rightarrow \text{First order system.}$$

$$k=R, \quad T=RA \rightarrow (T, k) \downarrow$$

$R \propto \frac{1}{\text{Area}}$ یا $R \propto \frac{1}{\text{Area}}$ $\left[\text{کل صافگی} \right]$ $\left[\text{بریدگی} \right]$ $R \propto \frac{1}{\text{Area}}$ $\left[\text{بریدگی} \right]$ $R \propto \frac{1}{\text{Area}}$ $\left[\text{بریدگی} \right]$

→ System described by 2nd order diff equation

$$\text{Transfer Function} \leftarrow \text{Laplace Transform} \quad a_0 \frac{d^2 y(t)}{dt^2} + a_1 \frac{dy(t)}{dt} + a_2 y(t) = b x(t)$$

$$\frac{a_0}{a_2} [s^2 y(s) - s \cancel{y'(0)} - \cancel{y(0)}] + \frac{a_1}{a_2} [s \cancel{y(s)} - \cancel{y(0)} + y(s)] = \frac{b}{a_2} x(s)$$

↓

$$\rightarrow \frac{a_0}{a_2} s^2 y(s) + \frac{a_1}{a_2} s y(s) + y(s) = \frac{b}{a_2} x(s)$$

$$\rightarrow y(s) \left[\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1 \right] = \frac{b}{a_2} x(s)$$

$$G(s) = \frac{y(s)}{x(s)} = \frac{b/a_2}{\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1}$$

↳ 2nd order #

$$\left(\frac{a_0}{a_2} \frac{d^2 y(t)}{dt^2} + \frac{a_1}{a_2} \frac{dy(t)}{dt} + a_2 y(t) = \frac{b}{a_2} x(t) \right) + a_2$$

$$\frac{a_0}{a_2} [s^2 y(s) - s \cancel{y(0)} - \cancel{y'(0)}] + \frac{a_1}{a_2} [s y(s) - \cancel{y(0)}] + y(s) = \frac{b}{a_2} x(s)$$

$$\frac{a_0}{a_2} s^2 y(s) + \frac{a_1}{a_2} s y(s) + y(s) = \frac{b}{a_2} x(s)$$

$$y(s) \left[\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1 \right] = \frac{b}{a_2} x(s)$$

$$G(s) = \frac{y(s)}{x(s)} = \frac{b/a_2}{\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1}$$

2nd order ← order of poles = poles - zeros

$$y(t)_{\infty} = \lim_{s \rightarrow 0} s y(s) = \lim_{s \rightarrow 0} s \left[\frac{k M}{s \tau^2 s^2 + 2 \zeta \tau s + 1} \right]$$

$$\Downarrow \quad y(t)_{\infty} = k M \quad \rightarrow \quad \frac{k}{\tau^2}$$

$$y(s) = \frac{k}{\tau^2 s^2 + 2 \zeta \tau s + 1}$$

$$y(t) = \mathcal{L}^{-1} y(s)$$

Partial Fraction

For quadratic equation $ax^2 + bx + c = 0$

$$x_{1,2} = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A}$$

$$\Rightarrow Ax^2 + bx + c = 0$$

$$\tau^2 s^2 + 2 \zeta \tau s + 1 = 0$$

$$s_{1,2} = \frac{-2 \zeta \tau \pm \sqrt{4 \tau^2 \zeta^2 - 4 \tau^2}}{2 \tau^2}$$

$$G(s) = \frac{y(s)}{x(s)} = \frac{b/a_2}{\tau^2 s^2 + 2\zeta\tau s + 1}$$

$$\frac{X(s)}{\tau^2} \left[\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1 \right] \rightarrow \left[\frac{k}{\tau^2 s^2 + 2\zeta\tau s + 1} \right] \quad \#$$

$\tau = \text{time constant}$ $\zeta = \text{damping factor}$

$\tau, \zeta \rightarrow$ dynamic behavior (part 1)

$k =$ steady state Gain
 $\zeta =$ damping factor

$k \rightarrow$ final steady state value (part 1)

most (dynamic behavior) is in p.s. system of p.p. controls

$$\begin{array}{c} X(s) \rightarrow \left[\frac{k}{\tau^2 s^2 + 2\zeta\tau s + 1} \right] \rightarrow y(s) \\ \text{IM} \rightarrow (\text{step change}) \end{array} \quad y(s) = \frac{k}{\tau^2 s^2 + 2\zeta\tau s + 1} \quad * \frac{M}{s}$$

Laplace inverse $\rightarrow y(t) = \lim_{s \rightarrow 0} s \left[\frac{kM}{s(\tau^2 s^2 + 2\zeta\tau s + 1)} \right]$

$$y(t)_{\infty} = kM \quad \#$$

$$\rightarrow [\tau^2 s^2 + 2\zeta\tau s + 1] \rightarrow \text{2-term 1st order}$$

$$Ay^2 + By + C = 0$$

$$s_{1,2} = \frac{-2\zeta\tau \pm \sqrt{4\zeta^2\tau^2 - 4\tau^2}}{2\tau^2}$$

$$s_{1,2} = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A}$$

$$\rightarrow \left[\frac{-\zeta \pm \sqrt{\zeta^2 - 1}}{\tau} \right] \rightarrow \frac{-\zeta}{\tau} \pm \frac{\sqrt{\zeta^2 - 1}}{\tau}$$

3 cases (1) $\zeta > 1$ (2) $\zeta < 1$ (3) $\zeta = 1$

I $\zeta > 1 \rightarrow$ 2 real different roots

II $\zeta < 1 \rightarrow$ 2 complex roots

III $\zeta = 1 \rightarrow$ 2 real equal roots

$$s_{1,2} = \frac{-\zeta \pm \sqrt{\zeta^2 - 1}}{T}$$

$$s_{1,2} = \frac{-\zeta}{T} \pm \frac{\sqrt{\zeta^2 - 1}}{T}$$

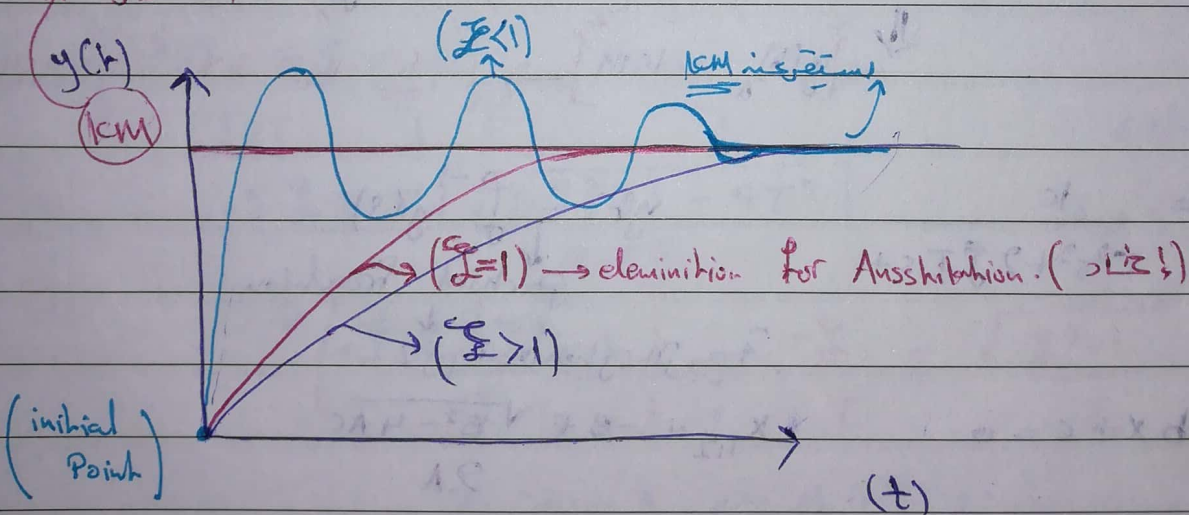
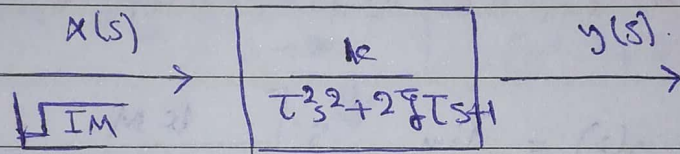
(Damping factor)

3 حالات ممكنة لـ ζ : $\zeta > 1$, $\zeta < 1$, $\zeta = 1$

- 1 $\zeta > 1$ → 2 real different roots → over Damped.
- 2 $\zeta < 1$ → 2 complex roots → under Damped
- 3 $\zeta = 1$ → 2 real equal roots → Critically Damped

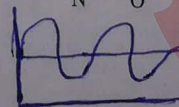
السلوك الدينامي (dynamic behavior) يعتمد على ζ و ω_n

التذبذب من الدرجة 1
Ausstellung



السلوك الدينامي (dynamic behavior) يعتمد على ζ و ω_n

cos و sin لـ $\zeta < 1$ و $\zeta = 1$ و $\zeta > 1$



$$y(t) = \dots \sin \dots \cos$$



(النمط)

* حيث الحد من Amplitude و تردد الحركات على
 في النظام و هو مستقر في $\frac{1}{\omega_n}$ ، قليل كان يتأرجح في $\frac{1}{\omega_n}$ و هو غير
 انما في Amplitude في الحركات في $\frac{1}{\omega_n}$ و هو مستقر

 ξ

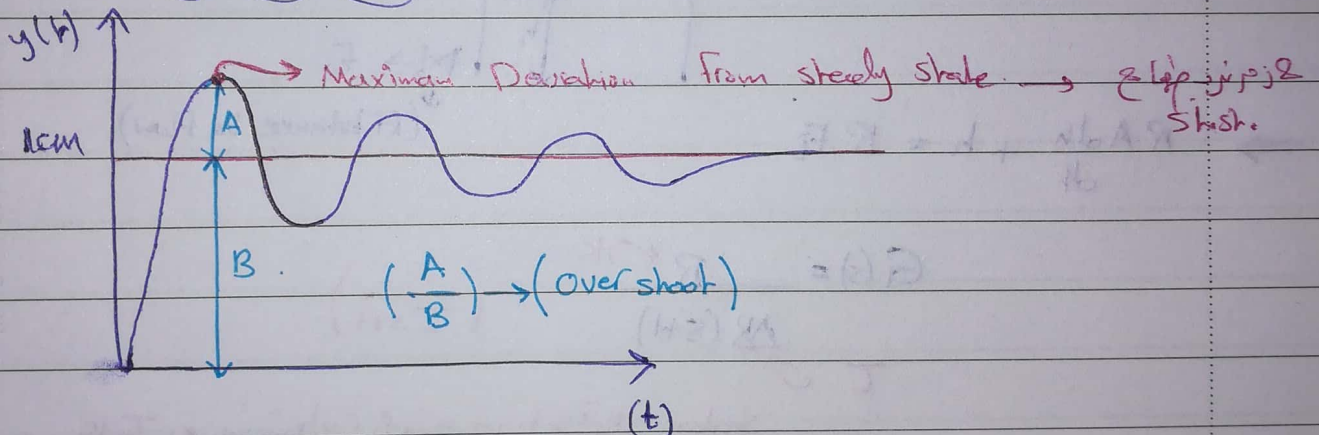
Damping Factor \rightarrow Parameter في Amplitude و هو

كل ما كان ξ كبير فقه ξ يكون في النظام

ξ \rightarrow قليل في حالة

down. or up ξ فقه

* Under Damped



$(\frac{A}{B}) \rightarrow$ Maximum Deviation from st. over steady state.

(ξ, τ, ω_n) \rightarrow Parameter system

Transfer Function \rightarrow Parameter

PP. parameter in Transfer Function \rightarrow (resonance)

Scanned by CamScanner

$$\left(\frac{a_0}{a_2} \frac{d^2 y(t)}{dt^2} + \frac{a_1}{a_2} \frac{dy(t)}{dt} + a_2 y(t) = \frac{b}{a_2} x(t) \right) + a_2$$

$$\frac{a_0}{a_2} [s^2 y(s) - s y(0) - \dot{y}(0)] + \frac{a_1}{a_2} [s y(s) - y(0)] + y(s) = \frac{b}{a_2} x(s)$$

$$\frac{a_0}{a_2} s^2 y(s) + \frac{a_1}{a_2} s y(s) + y(s) = \frac{b}{a_2} x(s)$$

$$y(s) \left[\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1 \right] = \frac{b}{a_2} x(s)$$

$$G(s) = \frac{y(s)}{x(s)} = \frac{b/a_2}{\frac{a_0}{a_2} s^2 + \frac{a_1}{a_2} s + 1}$$

2nd order \leftarrow order $n \Rightarrow p_1, p_2 = \text{poles}$

$$y(t)_{\infty} = \lim_{s \rightarrow 0} s y(s) = \lim_{s \rightarrow 0} s \left[\frac{1 \text{ cm}}{s^2 \tau^2 + 2 \zeta \tau s + 1} \right]$$

$$\Downarrow \left[y(t)_{\infty} = 1 \text{ cm} \right] \rightarrow \frac{1 \text{ cm}}{1} \text{ as } s \rightarrow 0$$

$$y(s) = \frac{1}{\tau^2 s^2 + 2 \zeta \tau s + 1}$$

$$y(t) = \mathcal{L}^{-1} y(s)$$

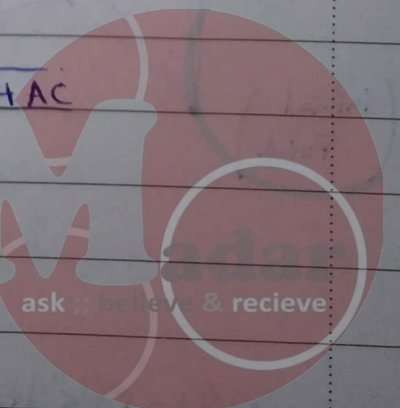
Partial Fraction

For $Ax^2 + Bx + C = 0$

$$x_{1,2} = \frac{-B \pm \sqrt{B^2 - 4AC}}{2A}$$

$$\tau^2 s^2 + 2 \zeta \tau s + 1 = 0$$

$$s_{1,2} = \frac{-2 \zeta \tau \pm \sqrt{4 \zeta^2 \tau^2 - 4 \tau^2}}{2 \tau^2}$$



$$s_{1,2} = \frac{-\zeta \pm \sqrt{\zeta^2 - 1}}{\tau}$$



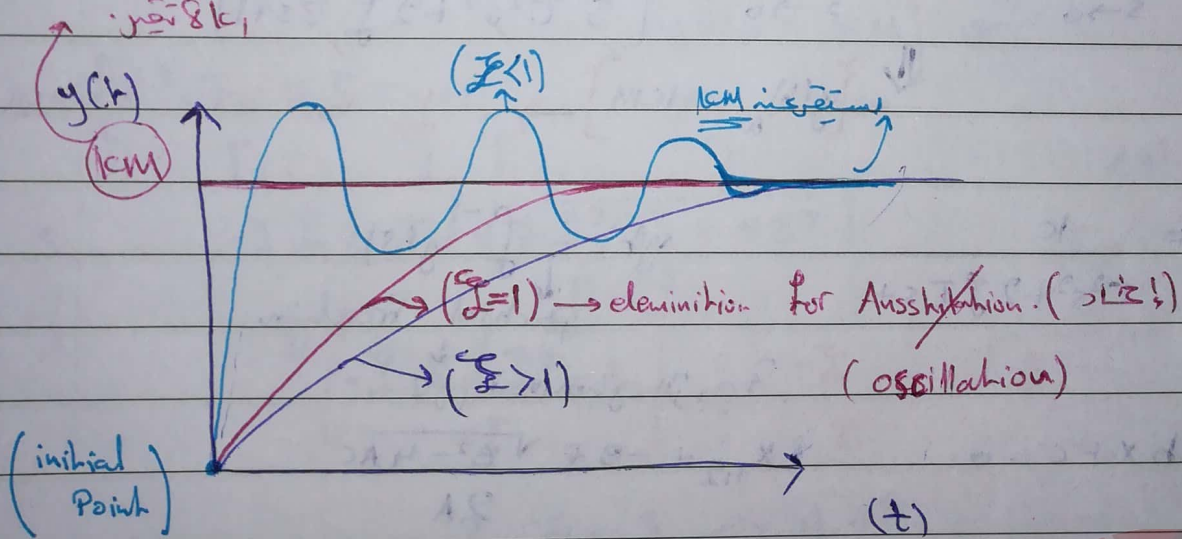
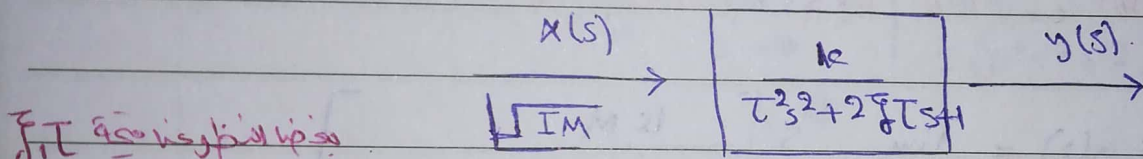
$$s_{1,2} = \frac{-\zeta}{\tau} \pm \frac{\sqrt{\zeta^2 - 1}}{\tau}$$

(Damping factor) ←

- 1] $\zeta > 1 \rightarrow$ 2 real different roots \rightarrow over Damped.
- 2] $\zeta < 1 \rightarrow$ 2 complex roots \rightarrow under Damped
- 3] $\zeta = 1 \rightarrow$ 2 real equal roots \rightarrow Critically Damped

dynamic behavior (dynamic behavior)

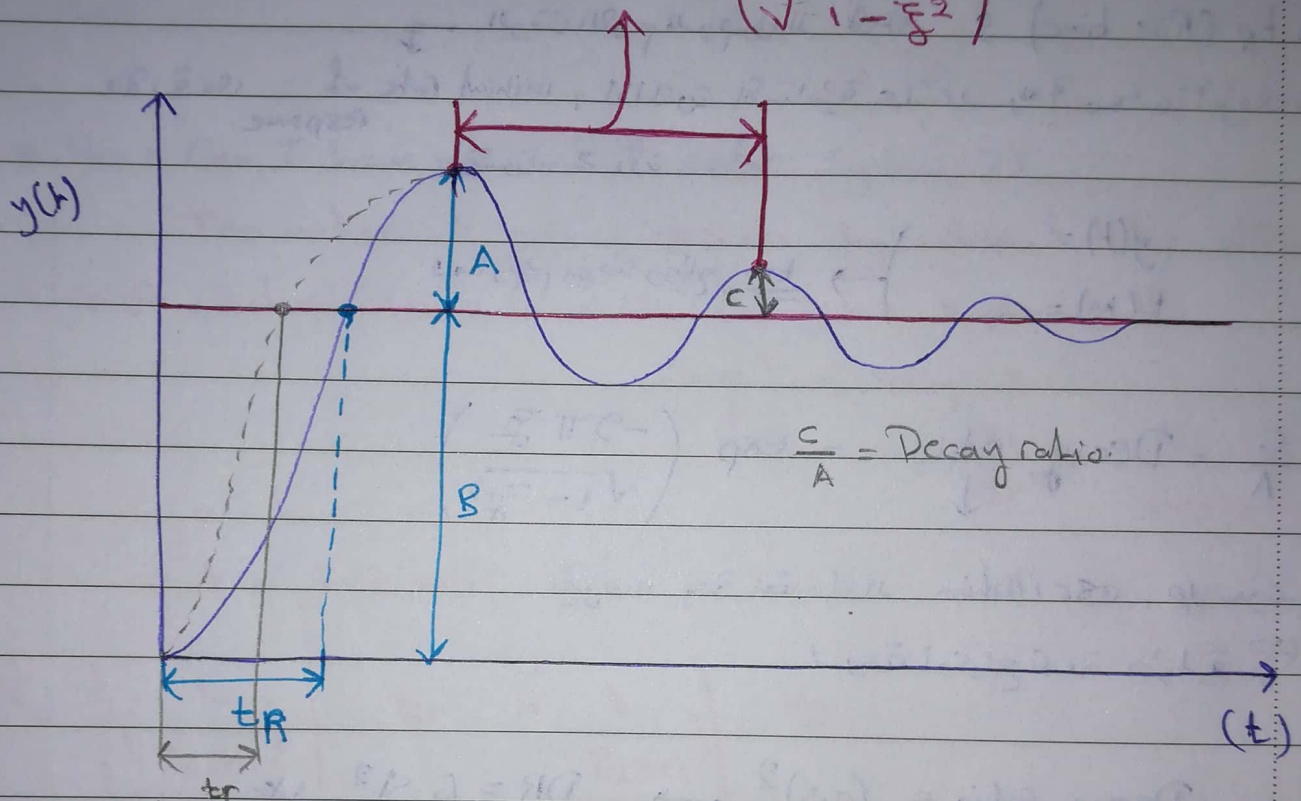
↓
Ausstellung



dynamic behavior (dynamic behavior) $y(t) = \dots \sin \dots \cos$

(هم لا نوسقلا عور ال time)

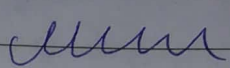
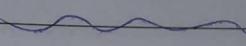
Period of oscillation $= P = \left(\frac{2\pi\tau}{\sqrt{1-\xi^2}} \right)$



* over shoot (O.S) = $\frac{A}{B} = \exp \left(\frac{-\pi \zeta \omega_n}{\sqrt{1-\zeta^2}} \right)$
 ↓
 maximum deviation. ω_n

* settling time $\approx \pm 5\%$ oscillation in st.st.
 \downarrow
 (st.st. \rightarrow approx $\omega \gg \omega_n$)

۱. ω \rightarrow frequency ω \rightarrow (ω) \rightarrow $\sin(\omega t)$
 ۲. A \rightarrow magnitude A \rightarrow $\sin(\omega t)$

Frequency
High Frequency \leftarrow 
Low Frequency \leftarrow 

* t_r (Rise time) \rightarrow الوقت الذي يستغرقه النظام للانتقال من 0 إلى 100% من القيمة المستهدفة.
initial rate of response.

$y(t) = \dots$
 $k(m) = \dots$ \rightarrow t_r \rightarrow time constant

* $\frac{C}{A} = \text{Decay ratio} = \exp\left(\frac{-2\pi\zeta}{\sqrt{1-\zeta^2}}\right)$

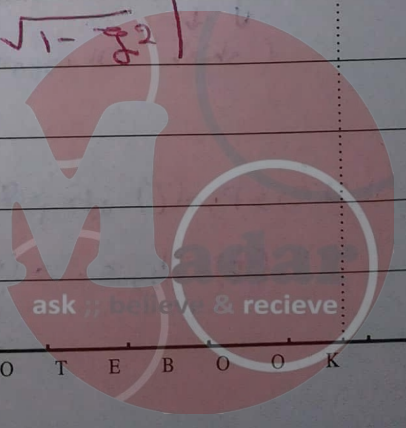
oscillation
PP

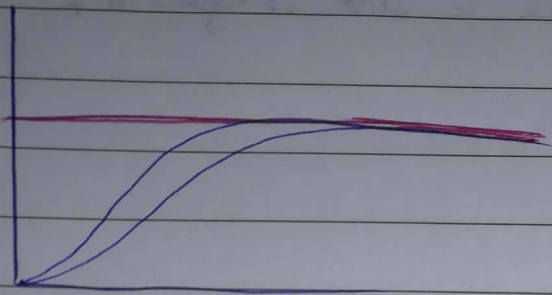
Decay ratio = $(0.5)^2 \rightarrow DR = (0.5)^2$ *

* design my control system.

* $(0.5) = 50\% = 0.5 \rightarrow 0.5 = \exp\left(\frac{-\pi\zeta}{\sqrt{1-\zeta^2}}\right)$

بما أن (0.5) هو (ζ)



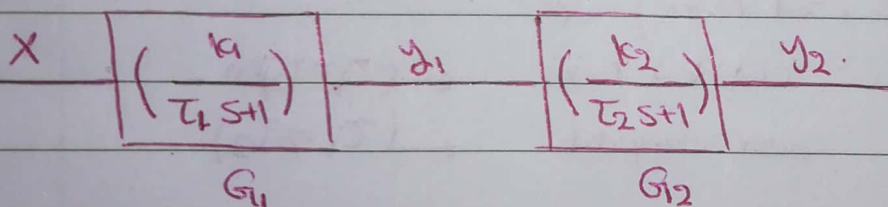


* How can I have a second order system ??

- The nature of system described by second order differential equation
 النظام ذو الترتيب الثاني يوصف بالمعادلة التفاضلية من الدرجة الثانية

Force \leftarrow Spring \downarrow
 acceleration = $\frac{d^2L}{dt^2}$ ← التسارع = المشتقة الثانية للموقع

- two first order system in series



$$G_1 = \frac{y_1(s)}{x(s)} = \frac{k_1}{T_1 s + 1}$$

$$G_2 = \frac{y_2}{y_1} = \frac{k_2}{T_2 s + 1}$$

لدينا y_2 كـ $f(x)$

$$\frac{y_2}{x} = \frac{y_1}{x} * \frac{y_2}{y_1} = \frac{y_2}{x} *$$

$$G = G_1 * G_2 = \frac{k_1 k_2}{(T_1 s + 1)(T_2 s + 1)} \quad \text{Second order system}$$

(overall G)

$$\left(\frac{k_1 k_2}{T_1 T_2 s^2 + (T_1 + T_2)s + 1} \right)$$

tip 2: 2 typical series \Rightarrow order of system \neq

$$G = \frac{k_1 k_2}{T_1 T_2 s^2 + (T_1 + T_2) s + 1}$$

$$\downarrow \quad \downarrow$$

$$T = \sqrt{T_1 T_2} \quad \#$$

$$\zeta = \frac{T_1 + T_2}{2\sqrt{T_1 T_2}} \quad \#$$

* why, can't be under Damped ??

Because my roots are real roots (T_1, T_2)

Complex \leftarrow roots \Rightarrow oscillate

(time constant) $\leftarrow \left(\frac{-1}{T_1}, \frac{-1}{T_2} \right)$ \Rightarrow time constant

time $\propto T_1, T_2$

* (I have real roots) *

critical roots

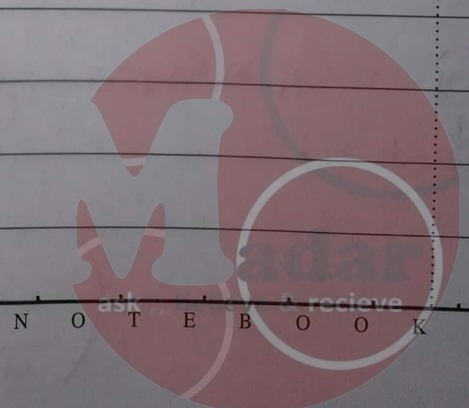
$\leftarrow (T_1 = T_2)$

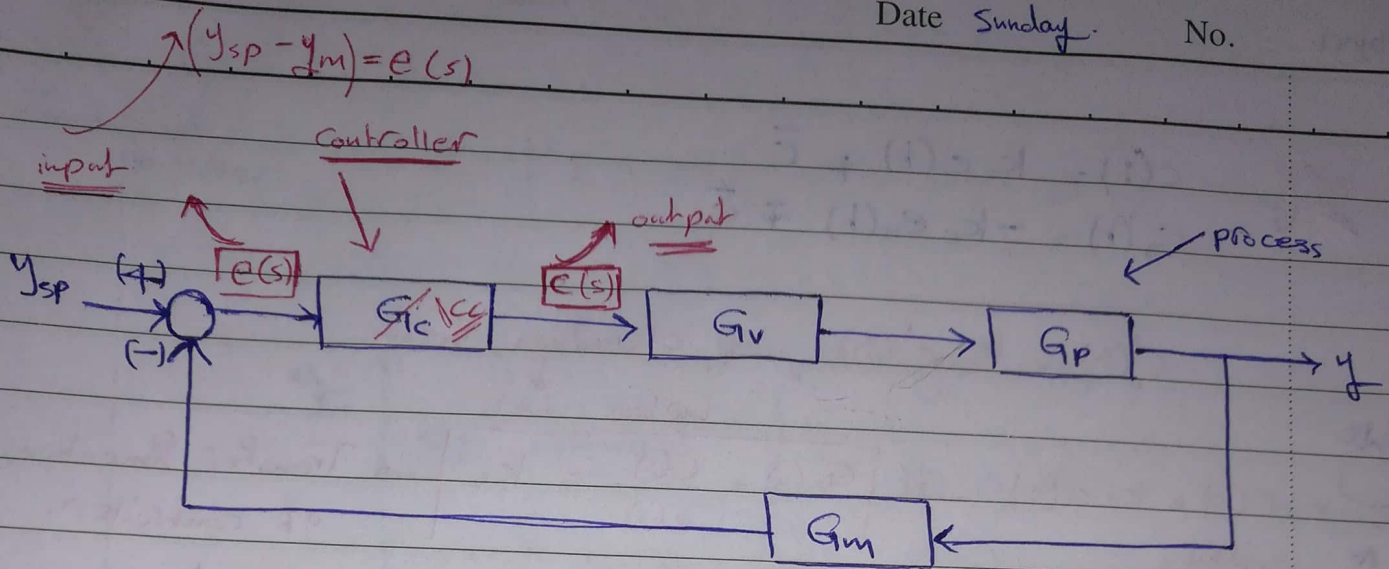
over damped

$\leftarrow (T_1 \neq T_2)$

... (...)

* chapter "5" done *





Transfer Function Block diagram

$$G_p: \frac{k}{Ts+1}$$

$$G_p: \frac{k}{T^2s^2 + 2\zeta Ts + 1}$$

Type of Controllers

Controller Transfer Function (controller input & output)

$$C(s) = G_c(s) * e(s)$$

$$C(t) = K_c e(t)$$

constant

output controller is directly proportional with $e(t)$ input of controller

(Self destructive Controller) \rightarrow if $e(t) = 0$, $C(t) = 0$

controller is this gain will go to valve \leftarrow No signal

Constant sh. sh.

$$e(t) = K_c e(t) + \bar{e}$$

(error = 0) deviation (deviation form)

steady state for controller $\rightarrow y_m = y_o$

$$c(t) = k_c e(t) + \bar{c}$$

$$c_{ss}(t) = -k_c e_{ss}(t) + \bar{c}$$

$$c'(t) = k_c e'(t) \rightarrow \text{deviation form.}$$

Laplace $s \rightarrow i$

$$C(s) = k_c E(s) \rightarrow \left| G_c(s) = \frac{C(s)}{E(s)} = k_c \right| \rightarrow \text{Transfer Function of controller.}$$

input is constant \rightarrow output = constant \rightarrow Transfer function \rightarrow constant \rightarrow is a \rightarrow

Because it has a zero dynamic \rightarrow is a action \rightarrow for controller.

System order \rightarrow is a order \rightarrow *

Zero order \rightarrow First order \rightarrow Second order \rightarrow *

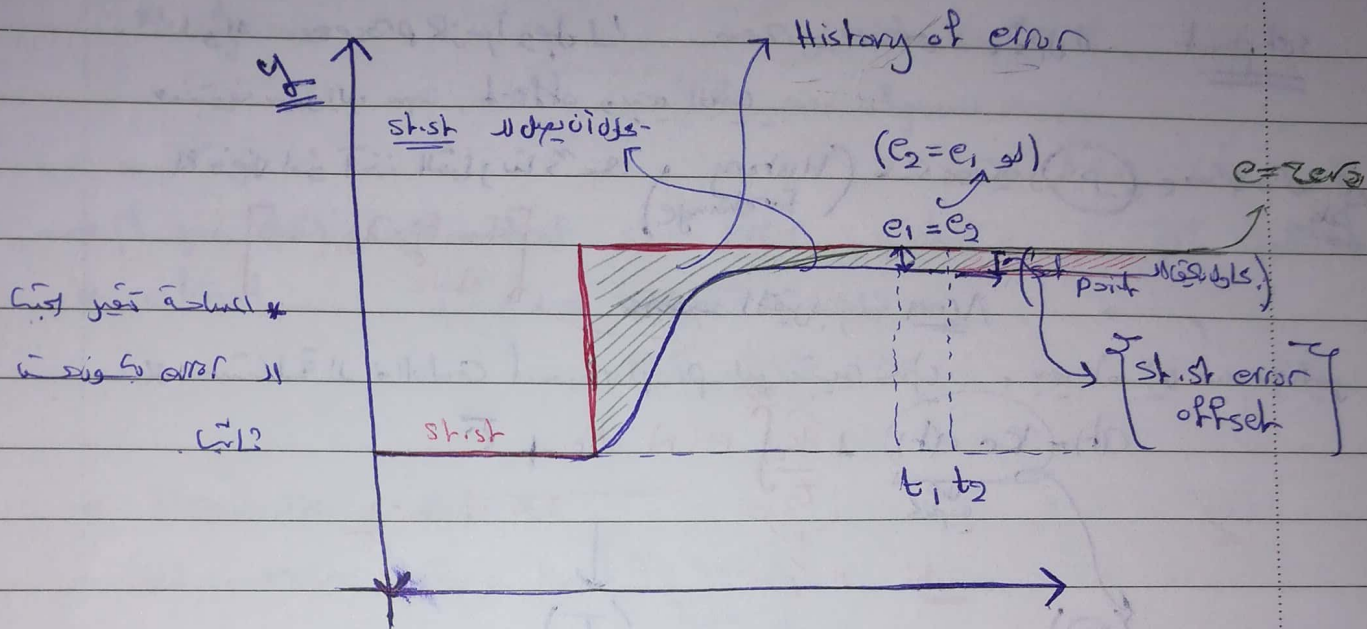
(spontaneous change) \rightarrow controller \rightarrow is a action

Controller \rightarrow is a action \rightarrow *

* Any process we not control manually, we can not control soft

Function of Controller.

→ اذا تم تغير بي اي اداة، اما تغير طابعها اي اداة



→ $G_1 = k_c e_1$

→ $G_2 = k_c e_2$

→ اذا تم تغير بي اي اداة، اما تغير طابعها اي اداة

→ اذا تم تغير بي اي اداة، اما تغير طابعها اي اداة

→ اذا تم تغير بي اي اداة، اما تغير طابعها اي اداة

sp difference between sp and shish, shish is dependent on output (steady state error (offset))

* The smallest change can detected by device → resolution

0.001 is resolution

0.001

0.0015

0.0014

* Function of the resolution for the device

Proportional Controller

NOT BOO K ask, believe & receive

Resolution of Controller

Resolution of process \rightarrow resolution of controller

set point \rightarrow error = zero \rightarrow process \rightarrow offset \rightarrow steady state

History for change \rightarrow (History for change)

Steadily changing state

Area

proportional controller \rightarrow proportional control

$$C(t) = K_c e(t) + \frac{K_c}{T_I} \int_0^t e(t) dt + \bar{C}$$

(P) (I)

[2]

Proportional Integral Controller (PI)

$$C(t) = K_c e(t) + \frac{K_c}{T_I} \int_0^t e(t) dt \rightarrow \text{deviation Form.}$$

$$C(s) = K_c e(s) + \frac{K_c}{T_I} \cdot \frac{1}{s} e(s)$$

$$G_c = \frac{C(s)}{E(s)} = K_c \left[1 + \frac{1}{T_I s} \right] = K_c \left[\frac{T_I s + 1}{T_I s} \right]$$

dynamics

$$G_c(p) = K_c$$

Simple

No dynamic

$$G_c(pI) = K_c \left[\frac{T_I s + 1}{T_I s} \right]$$

 T_I , K_c و s و 1 و T_I و s
 s و 1 و T_I و s
 s و 1 و T_I و s

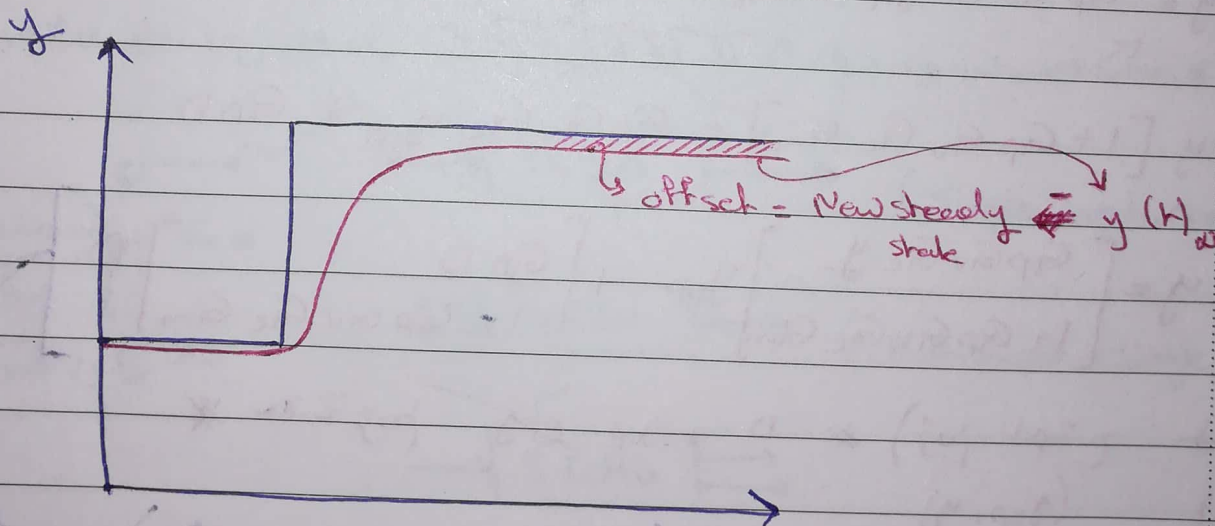
* if you don't care about steady error → don't go to PI controller.

* offset → d p process $y(t)$ و $y(t)$

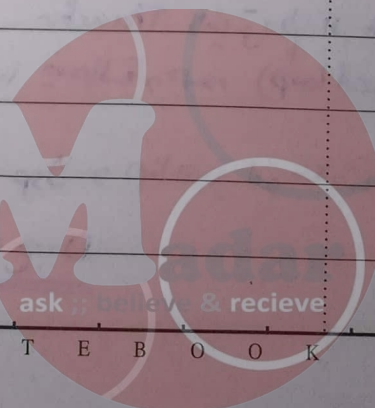
PI & PID

* P controller, $G_c = K_c$

* PI controller, $G_c = K_c \left[\frac{T_I s + 1}{T_I s} \right]$



offset = New steady state $y(t)$ steady state



* characteristic equation (Ch.E)

$$1 + G_c G_v G_p G_m = 0$$

System analysis is *
Ch.E is ch

Ch. system design Ch.E for the system

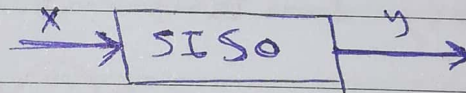
Transfer function is the ratio of output to input in the forward path \rightarrow (Transfer) \rightarrow output, input

loop is the ratio of output to input in the feedback path \rightarrow (1 + Transfer loop)



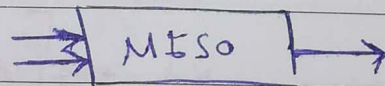
* SISO system *
SISO: Single input, single output

2 (D + y) is the ratio of output to input in the feedback path

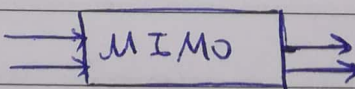
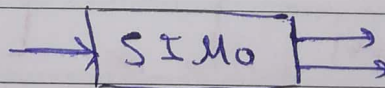


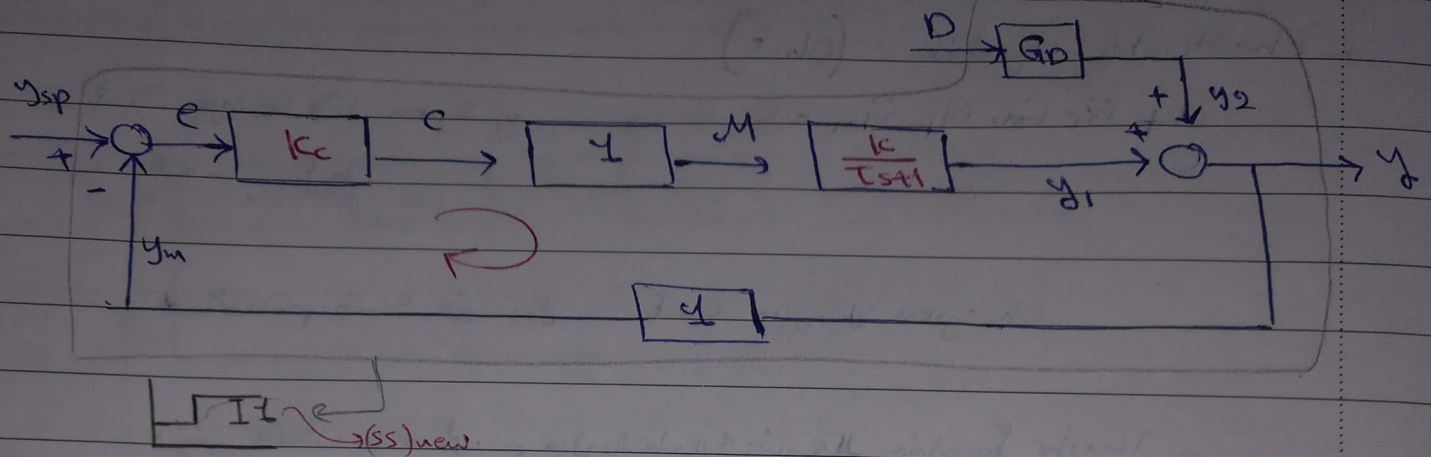
(Single input, single output)

(we have a change in the system)



(Multi input, single output)





$$y = \frac{K_c \left[\frac{k}{Ts+1} \right]}{1 + K_c \left[\frac{k}{Ts+1} \right]} * \frac{1}{s}$$

$$\text{offset} = \text{New SS} - y(t) \infty$$

(1) the magnitude ~~to the~~ change to the st. sh

~~$$y(s) = \frac{K_c \left[\frac{k}{Ts+1} \right]}{1 + K_c \left[\frac{k}{Ts+1} \right]} * \frac{1}{s}$$~~

$$y(s) = \frac{\frac{K_c k}{Ts+1}}{\left[\frac{Ts+1 + K_c k}{Ts+1} \right]} * \frac{1}{s} \Rightarrow y(s) = \frac{K_c k / (1 + K_c k)}{\frac{T}{Ts+1} + K_c k} * \frac{1}{s}$$

$$y(h)_\infty = \lim_{s \rightarrow 0} s \left[\frac{k_c k / (1 + k_c k)}{\frac{1}{1 + k_c k} s + 1} \right] \times \frac{1}{s}$$

$$y(h)_\infty = \frac{k_c k}{1 + k_c k} \quad \#$$

$$\text{offset} = 1 - y(h)_\infty$$

$$\text{offset} = 1 - \frac{k_c k}{1 + k_c k}$$

$$\Rightarrow \boxed{\text{offset} = \frac{1 + k_c k - k_c k}{1 + k_c k}} \quad \#$$

↓ is the offset is 1 \Rightarrow offset = $\frac{1}{1 + k_c k}$ $\#$

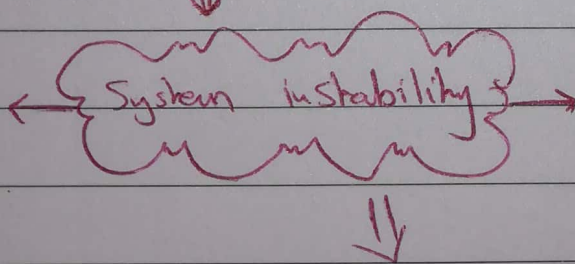
shesh → Control system diagram

* The offset inversely proportional with k_c . $\#$

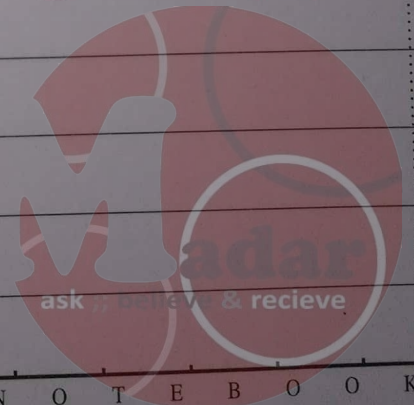
$$\left[\text{offset} \propto \frac{1}{k_c} \right] \rightarrow k_c \uparrow \Rightarrow \text{offset} \downarrow$$

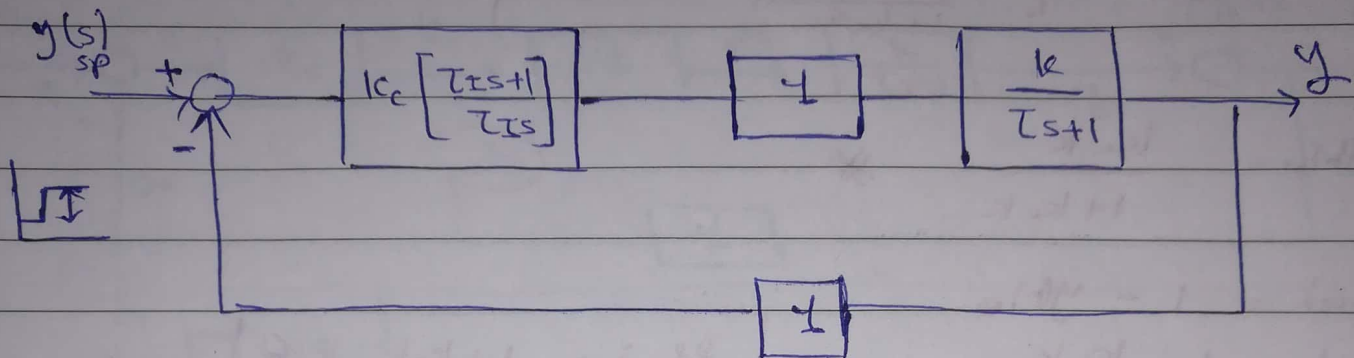
↓ is offset is inversely proportional to k_c

* There must be limit of $k_c \rightarrow$ (if k_c is too high, the system becomes unstable)



* offset is inversely proportional to k_c $\#$





$$\text{offset} = \frac{1}{1 + k_c k} \quad \#$$

$$y = \frac{k_c \left[\frac{\tau_I s + 1}{\tau_I s} \right] \left[\frac{k}{\tau_s + 1} \right]}{1 + k_c \left[\frac{\tau_I s + 1}{\tau_I s} \right] \left[\frac{k}{\tau_s + 1} \right]} \quad \# \quad \frac{1}{s}$$

no dynamic delay (PI) 1.
 Second order 1. First order

$$\text{offset} = \text{r.s.s} - y(t)_{\infty} \rightarrow$$

$$\text{offset} = 1 - y(t)_{\infty} \quad ??$$

$$y(s) = \frac{k_c k (\tau_I s + 1)}{\tau_I s (\tau_s + 1)} \cdot \frac{1}{s} \cdot \frac{1}{\left[\frac{\tau_I s (\tau_s + 1)}{\tau_I s (\tau_s + 1)} + k_c k (\tau_I s + 1) \right]}$$

$$y(t)_{\infty} = \lim_{s \rightarrow 0} s$$

$$y(t)_{\infty} = \frac{k_c k}{k_c k} = 1 \quad \#$$

Subject

Date

No.

$$\text{offset} = 1 - y(\infty)$$

$$= 1 - 1 = \text{Zero} \rightarrow$$

P(I) controller make elimination of offset *

* P(I) \rightarrow $K_c \rightarrow$ Gain

$T_I \rightarrow$ integral time.

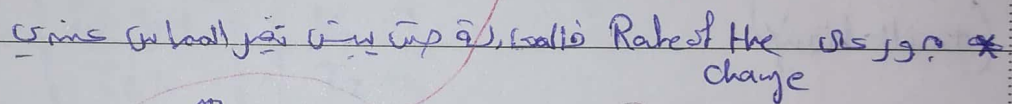
ديناميكي \leftarrow dynamic \leftarrow (S) plan \leftarrow (PI) \leftarrow ~~offset~~

\rightarrow The over all transfer function first order to second order

There possibility to ~~instillation~~ oscillation. (وحيث)

instillation \rightarrow ~~offset~~ \rightarrow (PI) *
(oscillation)





Ques Which is design element in JSX
Controller

In descriptive form.

$$C(s) = k_c e(s) + \frac{k_c}{T_I} \times \frac{1}{s} e(s) + k_c T_D s e(s) \rightarrow \text{Laplace}$$

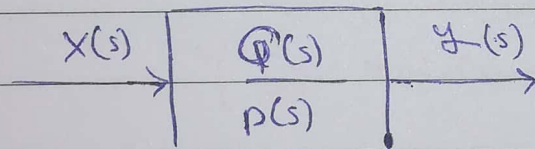
$$C(s) = k_c \left[1 + \frac{1}{T_I s} + T_D s \right] e(s) \quad (\text{PID controller transfer function})$$

$$G_c = \frac{C(s)}{e(s)} = k_c \left[\frac{T_I T_D s^2 + T_I s + 1}{T_I s} \right] \quad \text{Transfer Function}$$

(k_c, T_I, T_D) 3 parameters
offset بلایه ال

Sublimation of oscillation. (تبدیل امپلوس)

Physical unrealizability (عدم امکان پیاده سازی عملی)
(فیزیکی بودن و پیاده سازی)



$$\frac{Y}{X} = \frac{Q(s)}{P(s)} \Rightarrow Y P(s) = X Q(s)$$

output $P(s)$ $Q(s)$ input

physical unrealizability (ideal PID controller)

Physical unrealizability (عدم امکان پیاده سازی عملی)

$$\left[\frac{1}{s+1} \right] \quad \text{فیزیکی بودن}$$

Subject

Date

No.

Signal noisy



PID \rightarrow sensitive for noisy signal.

PI Controller $\rightarrow G_c = k_c \left[\frac{T_I s + 1}{T_I s} \right]$

P Controller $\rightarrow G_c = k_c$

PID Controller $\rightarrow G_c = k_c \left[\frac{T_I T_D s^2 + T_I s + 1}{T_I s} \right]$



$C = k_c E + \frac{k_c}{T_i} \int \dots \oplus k_c T_D \frac{de}{dt}$

(PI و PID کو PID سے کہتے ہیں)

میں نے

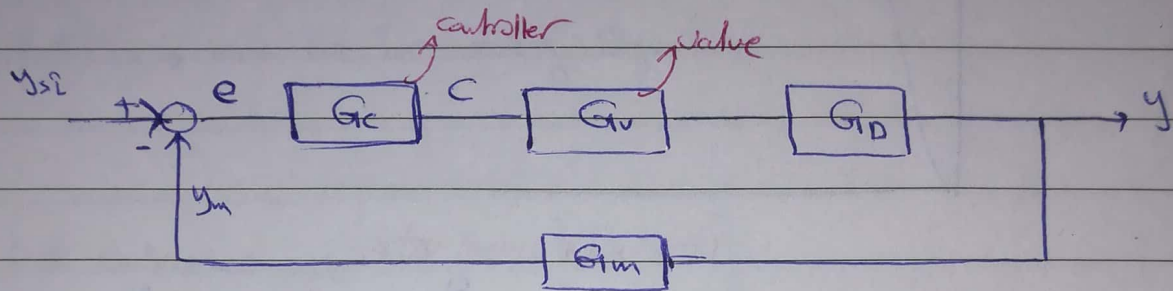
(TD ist das erste derivative term)

OK

* Controller Mode *

Controller output signal
process output signal

- 1 Direct Mode.
- 2 Reverse Mode.



* Direct Mode *

- if the output controller (C) increases when the controlled variable (y) increase, the Controller Mode is Direct.

Direct Mode: is a controller where (C) is ↓

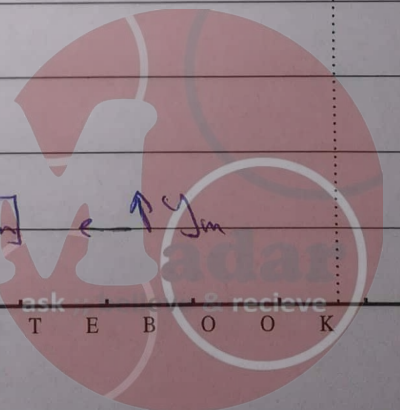
~~PP~~ Direct is a controller where C is ↓
derivative, integral, proportional is the action of the controller
... effect is positive

$$C = K_c [y_{sp} - y_m]$$

↑ ysp ↓ ym

[Kc < 0] is Kc is negative, C is ↓ [ysp - ym] is ↑ ym

* Direct Mode *



* Reverse Mode

- if the Controller output decreases when the Controller variable (y_m) increases, the Controller mode is Reverse Mode.

$$\downarrow C = K_c [y_{sp} - \uparrow y_m] \rightarrow [K_c > 0] \text{ Reverse Mode.}$$

Reverse or Direct if we are using Controller if not we are not

* Why we have to set the Controller Mode ??

valve. if we dip output of valve, valve will go to close - controller

→ To have (Synchronization) Between Controller and the valve.

if we dip output of valve, valve will go to close - controller

* Valve

Controller. if we dip output of valve, valve will go to close - controller

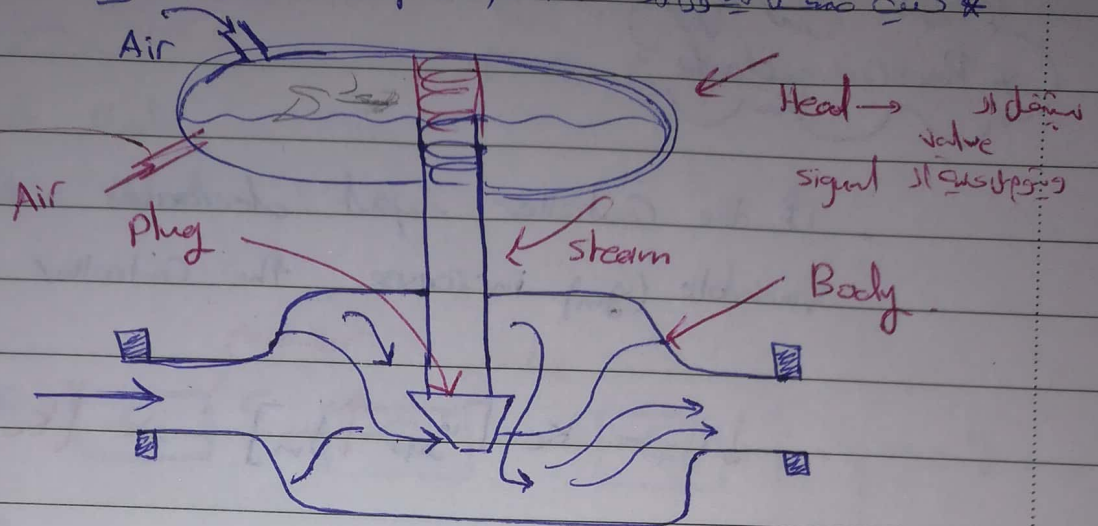
1] Fail open → if the controller fails, the valve open

2] Fail close → if " " " " " " close

if we dip signal to valve, valve will go to close (Air - To - close)

(Air - To - open)

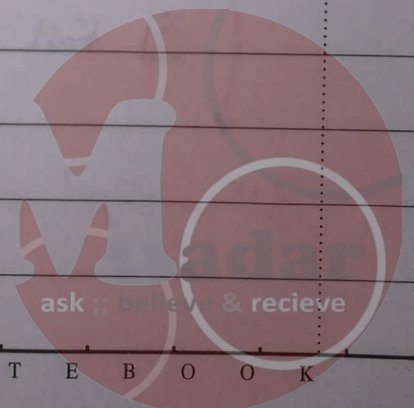
PE Air To close OR Air To open, valve



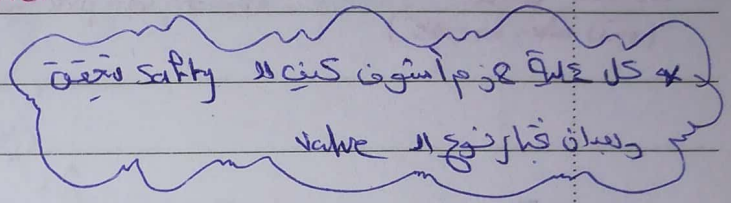
[Air To close → Fault open (Direct Mode)]

[Air To open → Fault close (Reverse Mode)]

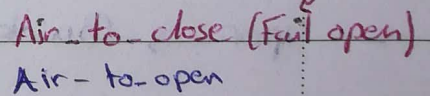
* why I have to choose OR my Controller output ??
one controller



- Safety of the process



* Q2: " " " " " " " " close Fully (Fail close design)



h \uparrow , signal of controller \downarrow , So the valve be Fail open
 ✖ (Reverse Mode) ✖
 (Air to close)

Control 11. 11. 2023

N O T E B O O K

* Flow characteristics of valves

* Flow characteristics of valves

Flow

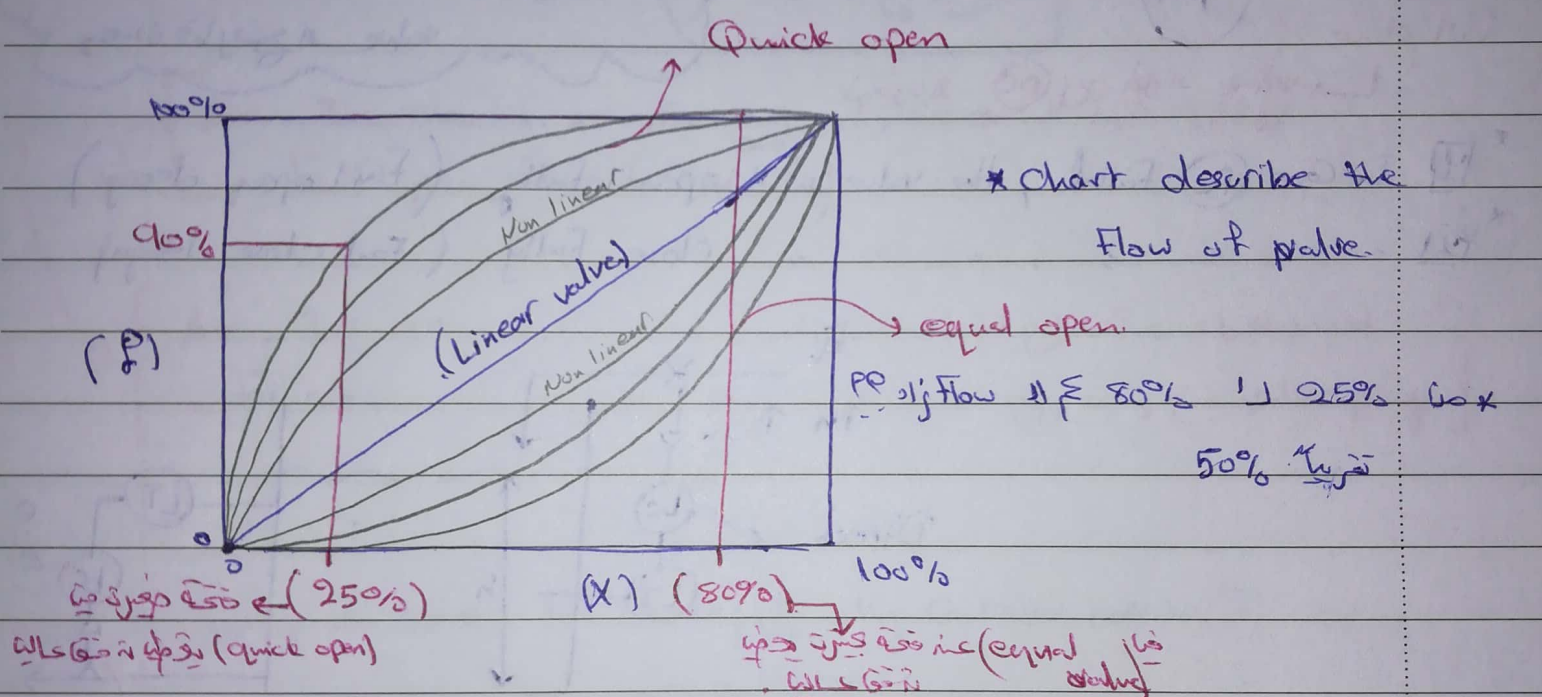
$$F = C_v \sqrt{\frac{\Delta P}{\rho}}$$

Driving Force (pressure drop across valve)

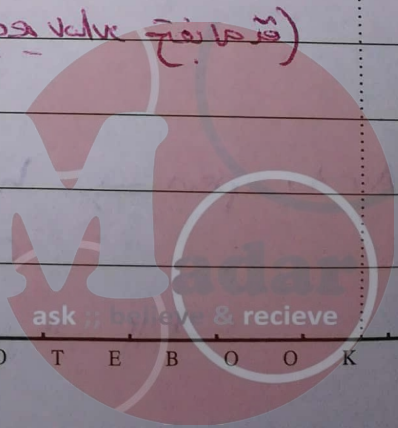
properties of fluid (density of fluid)

mechanical design of the valve

linear displacement of stem

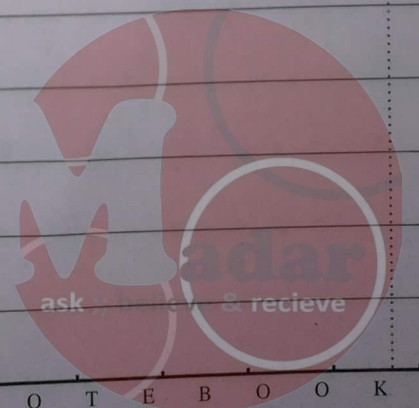
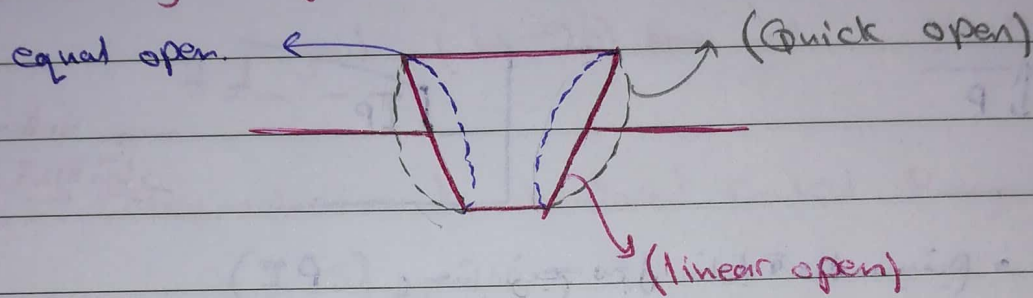


- upper → Quick opening → (Butt of flow)
- Below Linear valve → Equal opening (percentage) → (Butt of flow)
- linear valve → (Flow)
- * What to choose equal or Quick?



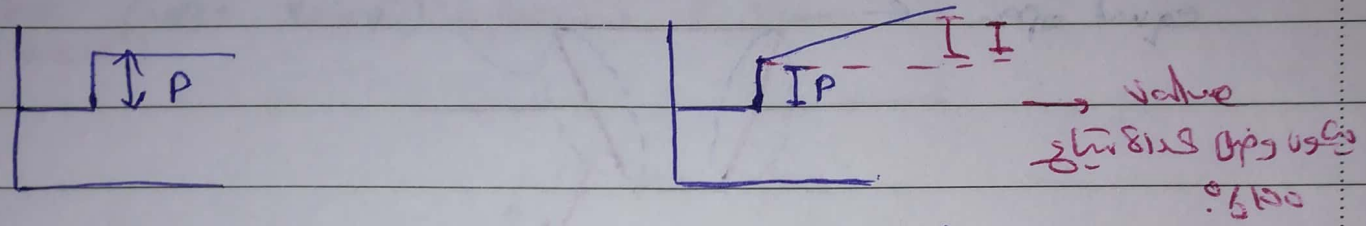
* Why I have to choose Linear OR Quick open OR equal percentage valve??

* How can be design the valve equal OR quick open??
By change the shape

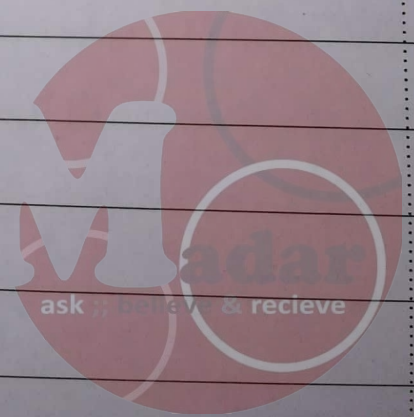


* wind up reset Controller \rightarrow PI controller
 \downarrow
 valve is closed controller is stopped

Which the set (PI) controller taken action, when the Controller take stagnation situation.



(PI) \rightarrow in this situation the controller will take action
 stagnation situation \rightarrow the error is zero



* Controller Signal *

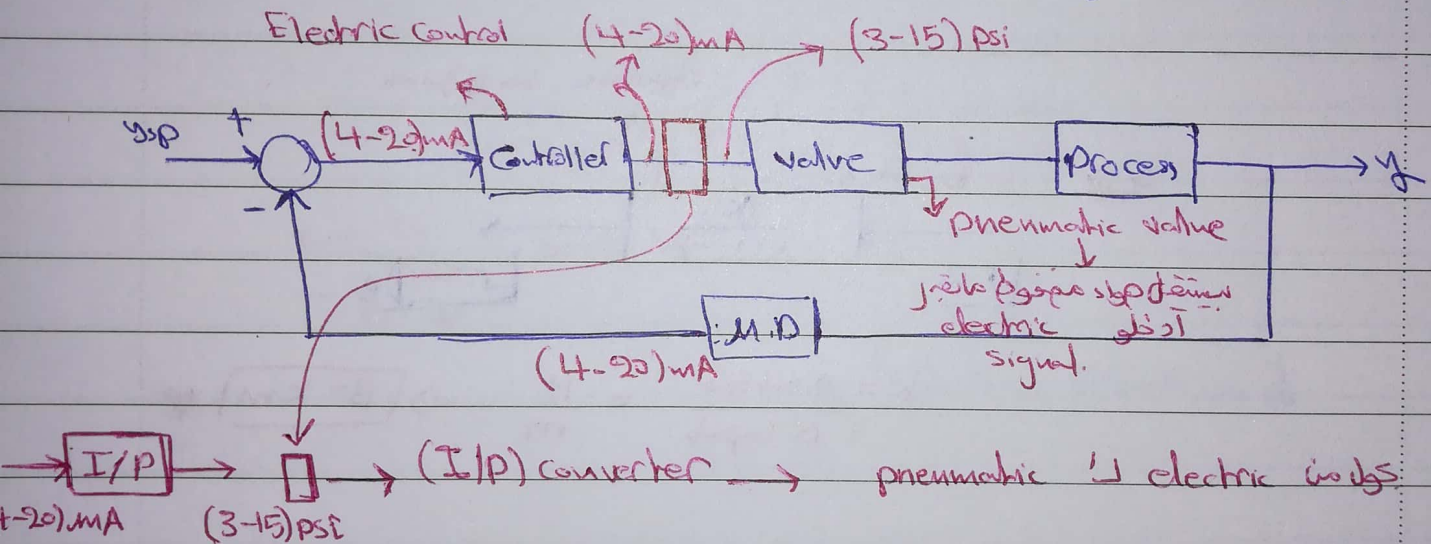
1 electric (أبجدي)

2 pneumatic (الهوائي) Compressed air pneumatic signal

The Electric and standard Control Range for signal (4-20) mA (4-20) mA *

Pneumatic and standard Control Range signal (3-15) psi *

PP pneumatic & Electric في هذا المثال *



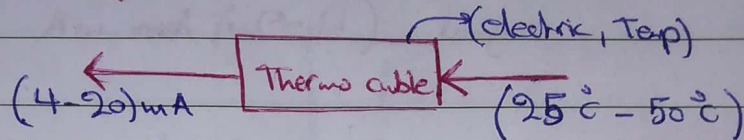
* thermo cable → electro motive Force

electric signal is converted to pneumatic signal
pneumatic signal is converted to electric signal

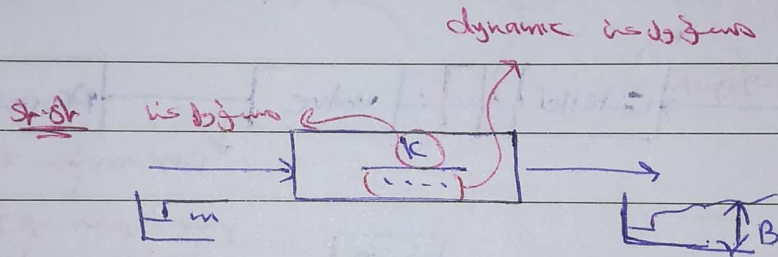
* What the Benefit to having a standard signal??

① Standard design is easier to design
standard design

② Standard design is easier to design



50°C → 20 mA, 25°C → 4 mA
(20 mA) output at 50°C
(4 mA) " " " " 25°C



$$k = \frac{\text{Output}}{\Delta \text{input}} = \frac{B}{m} \Rightarrow \boxed{B = km}$$

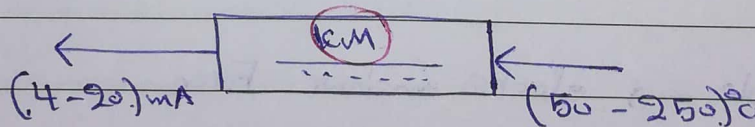
Standard gain

input

output

Gain

Static Gain



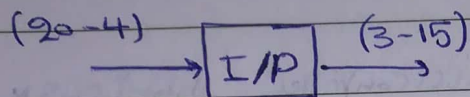
$$km = \frac{\text{Output}}{\Delta \text{input}} = \frac{20-40}{250-50} = \frac{16}{200}$$

$$\boxed{km = 0.08}$$

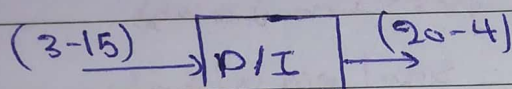
Given I/P, K is used to be calibrated WLS 1/3

4 mA \rightarrow 50°C

20 mA \rightarrow 250°C

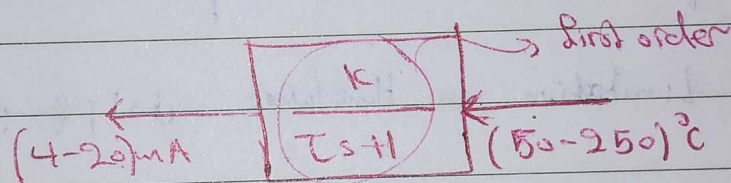


$$K = \frac{15-3}{20-4} = \frac{12}{16} \quad \#$$



$$K = \frac{16}{12} \quad \#$$

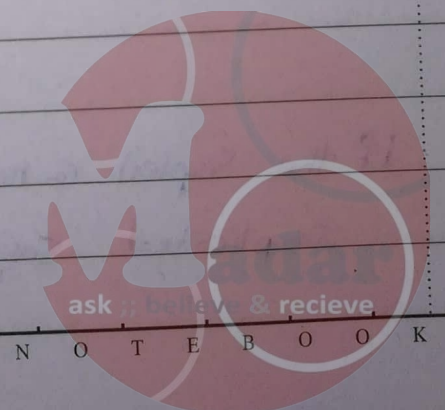
* Thermocouple a first order with time constant equal 5, calibrated to 50-250°C with electric control.

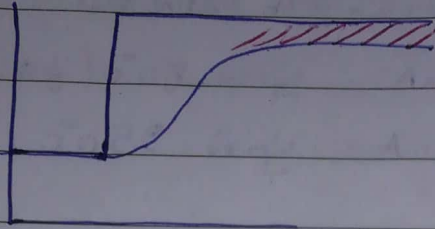


$$K = \frac{16}{200} = 0.08 \quad \#$$

$$T = 5 \quad \#$$

K is used to be calibrated



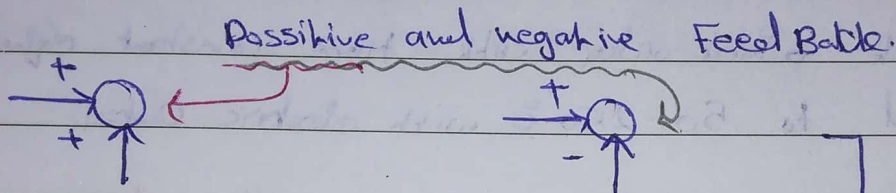


∞ Control Controller

PI system offset

P

(PI) PID e - Signal noisy



Limitation on Hardware

Setpoint
change in disturbances

PP possible

PP when offset

PP value

no gain like position

Explain

$$y(t) = KM - \left[\frac{\omega}{\sqrt{1-\zeta^2}} \sin(\omega t + \phi) \right]$$

Simplest. Control

if the Simplest control give what i need, not need to complicated system

Simple

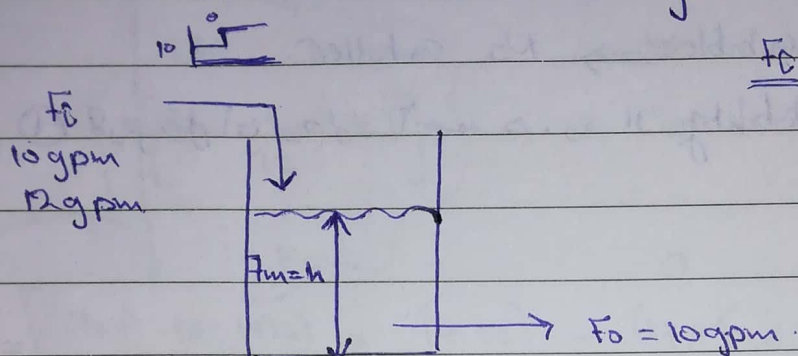
no offset

Task E, B, O & Ciele

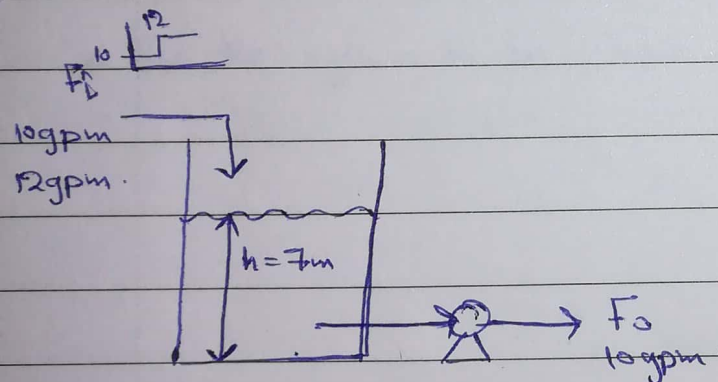
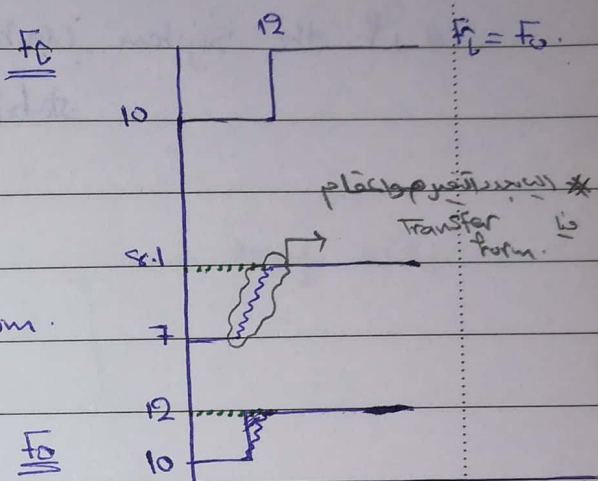
* Control System Performance

- stability of control system

- what is the stability?

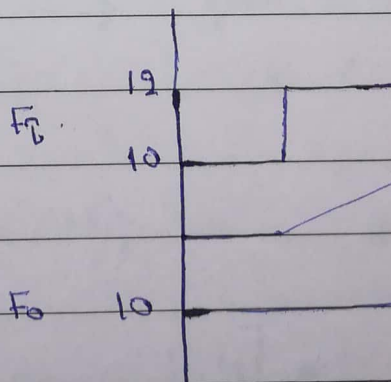


@ SS: h constant.



@ SS: h constant.

$$F_d = F_o = 10 \text{ gpm.}$$



⇒ system will Analysis using
the 11 g. Transfer function
(s-01)

What is the stability? if the output is bounded (or not) is due to bounded input → The system is stable

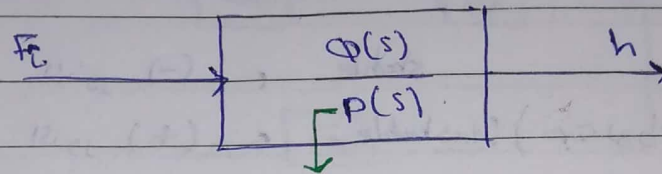
* if the output is unbounded is due to bounded input \Rightarrow The system

Subject

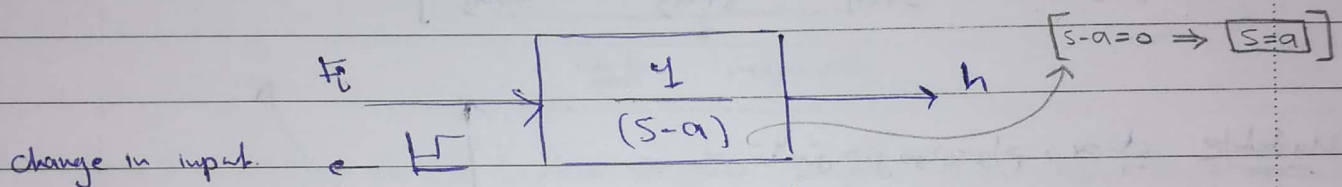
Date

No. Unstable *

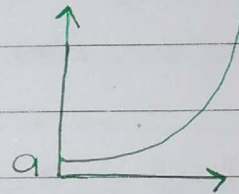
Transfer function of process is given by
output is ∞ output is ∞



characteristic equation is $s = -a$

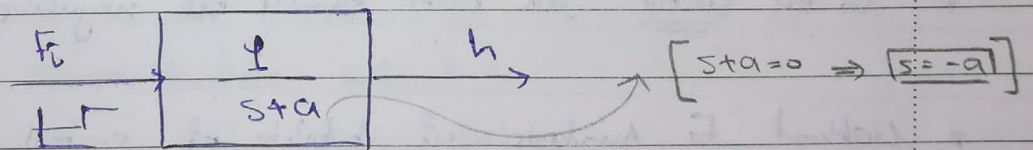


$$\mathcal{L}^{-1}\left(\frac{1}{s-a}\right) = e^{at}$$

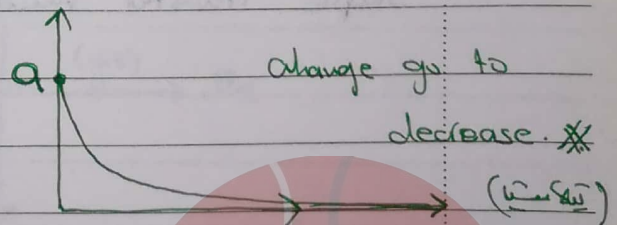


The change goes to increase to ∞ (unstable)

* Unstable System *



$$\mathcal{L}^{-1}\left(\frac{1}{s+a}\right) = e^{-at}$$



change goes to decrease *

* (stable system) steady state is reached

ask, believe & receive

∴ stability $\propto \frac{1}{\Delta E}$ ✓

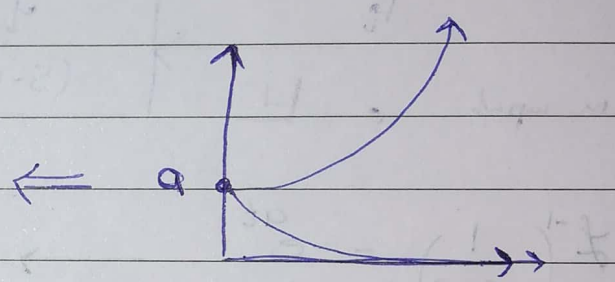
131 Transfer Function \rightarrow post jo de gya.

stable (-) الجذور

اُستور (+) ← Unstable (مستقر و لا negative) Unstable

$$\mathcal{L}\left[\frac{1}{s-a} * \frac{1}{s+a_1} * \frac{1}{s+a_2} * \frac{1}{s+a_3}\right]$$

Unstable تائیں ہیں جو پلاسٹک ہے



* if one of the roots positive the system Unstable

* To Be stable All roots should be negative.

* Method For Analysis of stability of control System

Simple Transfer Function \rightarrow closed loop Transfer Function

* Stability Analysis *

characteristic Equation

- Roots

Root Locus Method

one parameter controller

$$1 + G_c G_p G_m = \text{Zero}$$

$$5s^4 + 7s^3 + \dots = \text{Zero}$$

الجزء ١ من الحل

(5) الحل الثاني

controller

Parameter

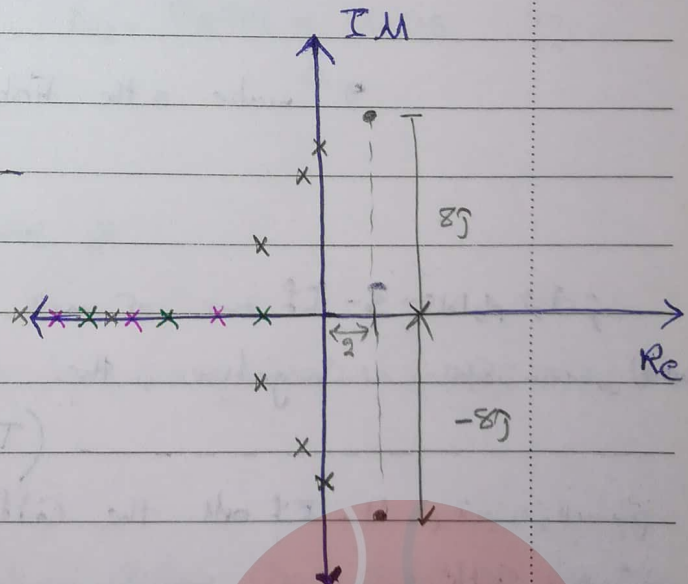
المعاملات التي تتغير مع تغير المعاملات

$$9s^3 + 7s^2 + 11s + 1 + k_c = 0$$

الحل ٢ من الجزء ١

k_c هو المعامل

| k_c | R_1 | R_2 | R_3 |
|---------------|-------|---------------|-------------|
| 0 | -3 | -5 | -7 |
| 10 | -3.7 | -7 | -9.5 |
| 20 | -9 | $-2 \pm 3j$ | $-2 - 3j$ |
| 50 | -11 | $-0.5 \pm 7j$ | $-0.5 - 7j$ |
| critical ← 55 | -13 | $+7.5j$ | $-7.5j$ |
| 57 | -3 | $+2 + 8j$ | $+2 - 8j$ |



→ The system is stable (negative roots).

الحل ٢ من الجزء ١ (Imaginary roots) ←

unstable → stable

NOT BOOK

Critical Stable

* نوعی PI أو PID مع تغير كبير في المدخلات (K_c, T_i, T_d) بحسب العلاقة أعلاه

[2] Routh - Hurwitz Array

$$\downarrow \quad \text{of } \underline{s} \quad \left[a_0 s^n + a_1 s^{n-1} + a_2 s^{n-2} + a_3 s^{n-3} + \dots + a_n s^0 = 0 \right]$$

coefficients

2. make the first ~~element~~ positive [absolute value
negative value]

3- If one or more of the coefficients (a 's) is/are negative, the system Unstable.

(Test I)

4. If all the coefficients are positive \Rightarrow Go To Test II

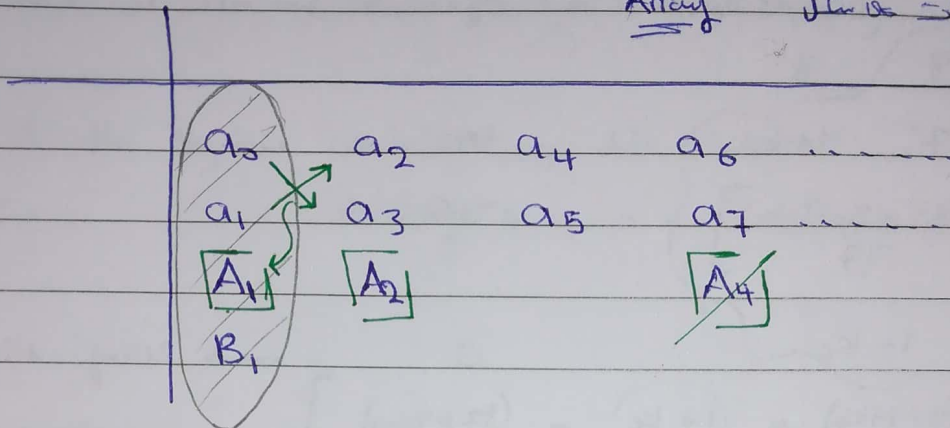
group stable

FALSE

⇒ Test II :-

* Form the R-H Array *

Array \Rightarrow $\frac{a_1}{a_0}$



$$A_1 = \frac{a_1 a_2 - a_0 a_3}{a_1} \quad \checkmark$$

$$A_2 = \frac{a_1 a_4 - a_0 a_5}{a_1} \quad ??$$

$$A_4 = \frac{a_1 (zero) - a_0 (zero)}{a_1} \Rightarrow \text{zero} \quad \times$$



* For the system stable \Rightarrow all coefficient in Column One should be positive if any one is negative the system is unstable.

Ex: $8s^3 + 17s^2 + 3s + 1 + k_c = 0$ (5°)

$$\begin{array}{cc} 8 & 3 \\ 17 & 1+k_c \\ \hline \frac{17 \times 3 - 8 \times (1+k_c)}{17} & \\ \hline 1+k_c & 0 \end{array}$$

$$B_1 = \left[\frac{(17 \times 3) - (8 \times (1+k_c))}{17} \times (1+k_c) - (17 \times 0) \right]$$

$$\left[\frac{(17 \times 3) - (8(1+k_c))}{17} \right]$$

* For the system to be stable $\Rightarrow \frac{17 \times 3 - 8(1+k_c)}{17} > 0$ positive

$$\left[\begin{array}{l} 1+k_c > 0 \\ (k_c > -1) \end{array} \right] \text{ positive.}$$

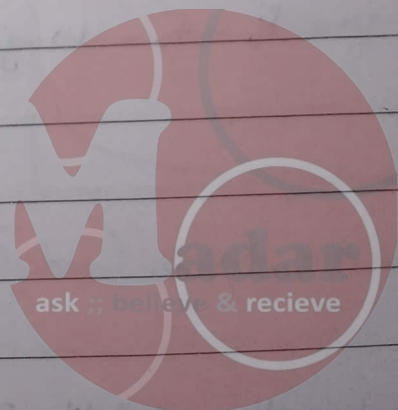
$$\frac{17 \times 3 - 8(1+k_c)}{17} > 0 \rightarrow 17 \times 3 < 8(1+k_c) - 1$$
$$k_c < 5.4$$

$$1+k_c > 0 \rightarrow (k_c > -1)$$

* $[-1 < k_c < 5.4]$ * stability Range

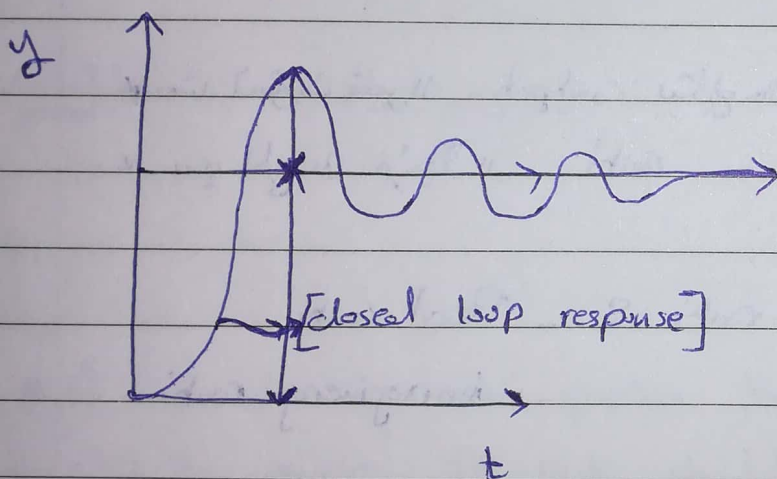
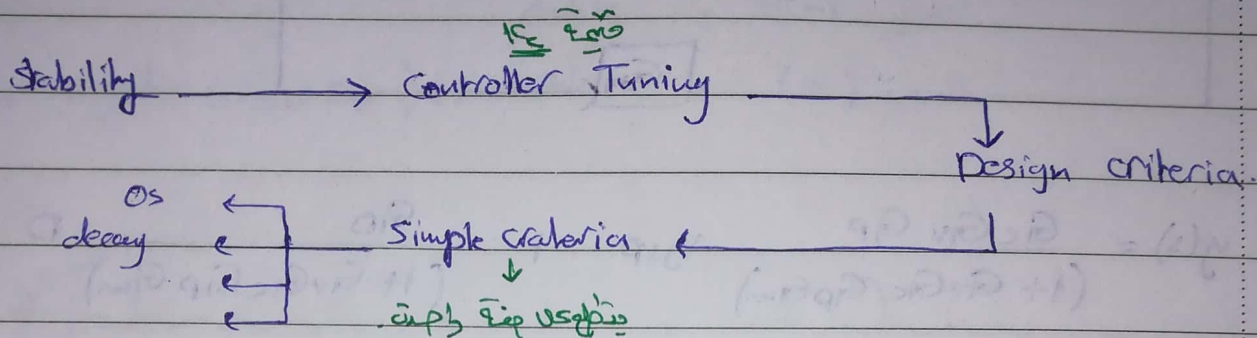
k_c is gain. $\frac{1}{1+k_c k}$

① لازم قیاسی و دلالتی استوار و دھندہ الی stability.

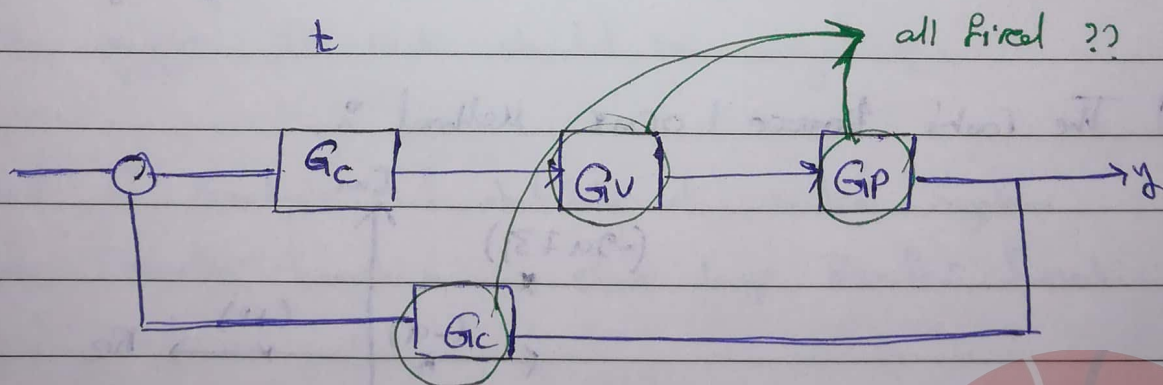


* Design of Control System Stability *

PP Kc 1/200 jian gawes, Kc 1/20 Range 1/200 1/200



Controller Parameter response



Response. No. ~~fixed~~ p. ~~fixed~~ decay, OS No. ~~fixed~~

* What is the value of Controller Gain?
which Give $0.5 < 0.5$ (over shoot)

* Over shoot express about the Maximum deviation from Set
* The criteria may be decay ratio or

Controller \rightarrow $\frac{y}{u}$ \rightarrow $\frac{y}{u}$ \rightarrow $\frac{y}{u}$

* Criteria

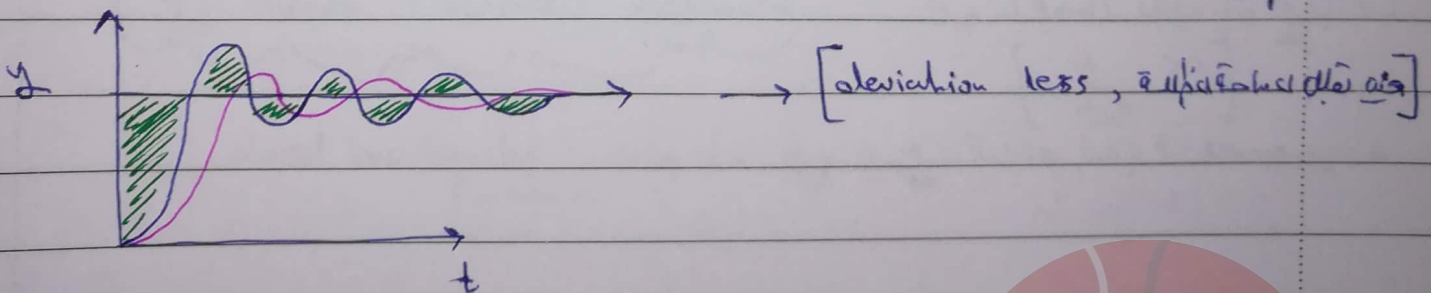
To minimize the over shoot and minimize the rise time.

$$OS = \exp \left[\frac{-2\pi}{\sqrt{1-\zeta^2}} \right]$$

\rightarrow No controller with these 2 condition. Because they are inversely

Comprehensive Criteria

Minimize the deviation as ~~overall~~ overall between setpoint and Response.



* Target of controller \rightarrow To minimize the deviation between the setpoint and Response.

* * Minimize deviation \rightarrow reduce the Area under the curve. *

Ex 10 Calculate the Area

~~2~~ flip sign -ve, +ve, same

قوله لا اله الا الله

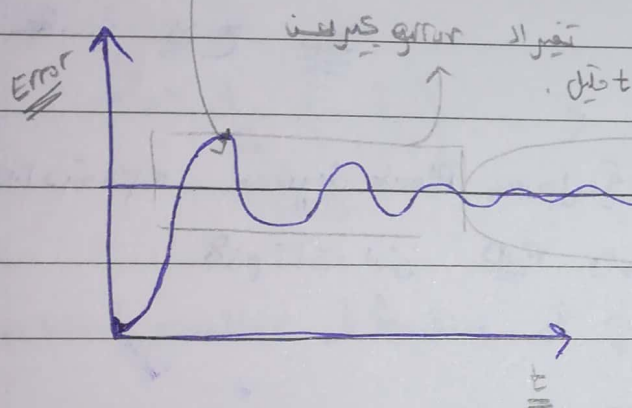
$$\rightarrow IAE = \int_0^{\infty} |e(t)| dt$$

* Comprehensive Design Criteria (allows derivation of optimum)

1 Integral Method

$$\textcircled{1} \quad I_{SE} = \int_a^b e^2 dt$$

(2) $IAE = \int_0^{\infty} |e| dt$



$$(3) \text{ ITAE} = \int_0^{\infty} t |e| dt$$

* استخدام طريقة ① ② ③ ، باعتماد على وفور Controller.

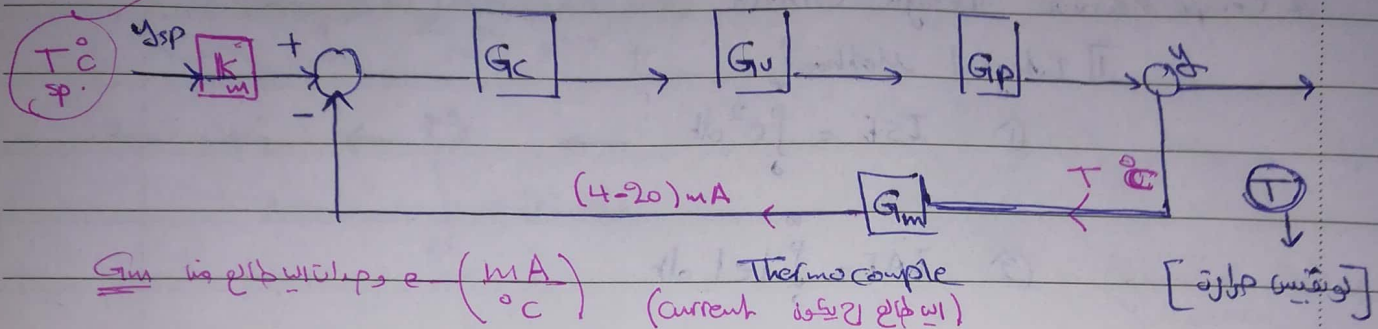
2 Direct Synthesis Method

کیت و بی ایلم با المینا' سپ

$$\left[\frac{y}{y_{sp}} \right]$$

closed loop transfer function is $\frac{Y}{U}$ and $\frac{Y}{U}$ is the closed loop transfer function.

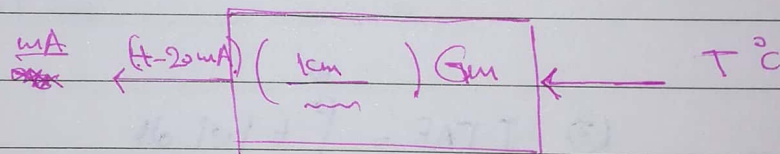
subject



$$y = \frac{G_c G_v G_p}{1 + G_c G_v G_p G_m} y_{sp}$$

$\frac{y_{SD}}{\sqrt{n}}$ \xrightarrow{d} $N(0, \sigma^2)$ *

Design is drawing, qualitative implies Block diagram & flowchart
units & implies quantitative is B.D. & strip



$$k_{em} = \frac{\Delta \text{output}}{\Delta \text{input}} = \frac{mA}{\mu e}$$

* let $G_{uv} = K_{uv}$ and let $G = G_u G_p G_v$

منهج البحث: البحث ذو الأسس النظرية، وأما عن
Controller

$$y = \frac{G G_c}{1 + G_c G} y_{sp}$$

$$G_c = \frac{1}{G} \left[\frac{y|y_{sp}}{1 - y|y_{sp}} \right]$$

(Direct Synthesis Method)

(y/y_{sp}) is the GC is the GC

(Performance criteria)

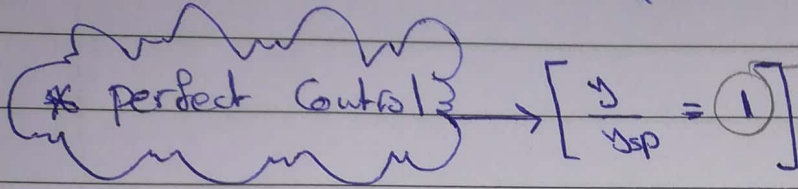
ask : believe & recieve

\Rightarrow you have relationship between GICs of system and performance criteria and G.

spec G_c \rightarrow $\frac{y}{y_{sp}}$ (y/y_{sp}) \rightarrow $\frac{y}{y_{sp}} = 1$ performance \rightarrow $\frac{y}{y_{sp}} = 1$

Direct

Transfer function with zero order



not achieve because of

$$G_c = \frac{1}{G} \left(\frac{1}{1 \times 1} \right) \rightarrow$$



ممكن؟

[الاولى و الثانية]



Final??

* what another definition of perfect control ??

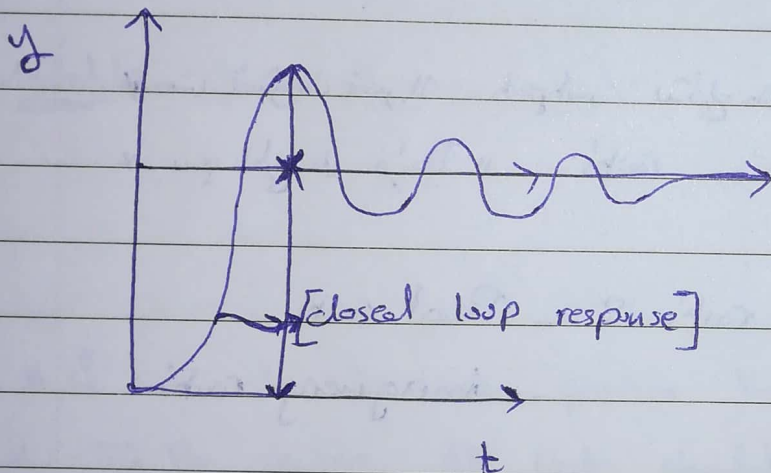
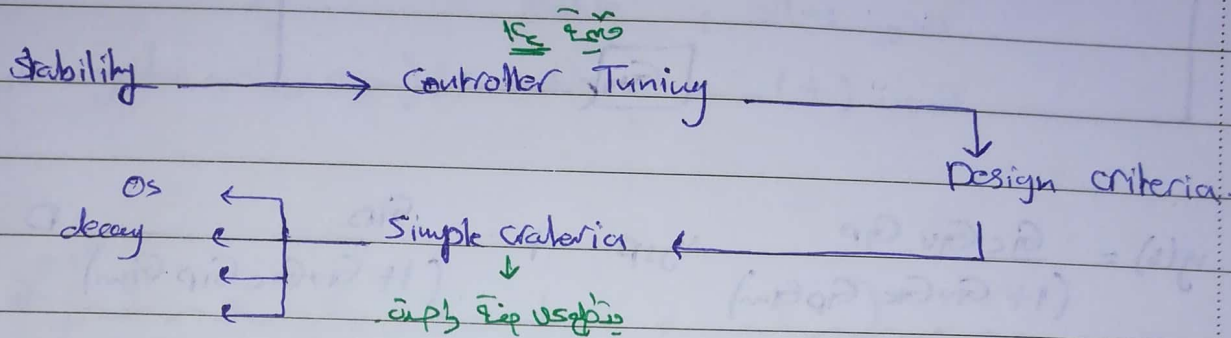
$$\left[\frac{y}{y_{sp}} = \frac{1}{0} = \infty \right] \text{ Zero order process } \rightarrow \left[\frac{y}{y_{sp}} = 1 \right] \text{ via } *$$

$$\left[\frac{y}{y_{sp}} = \frac{1}{Ts+1} \right] \rightarrow \text{(First order)} \rightarrow$$

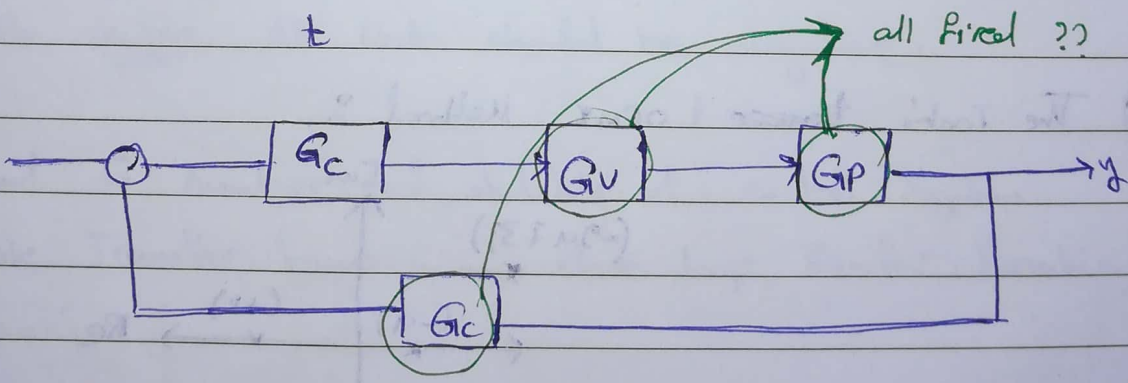
next step \rightarrow Zero order

* Design of Control System Stability

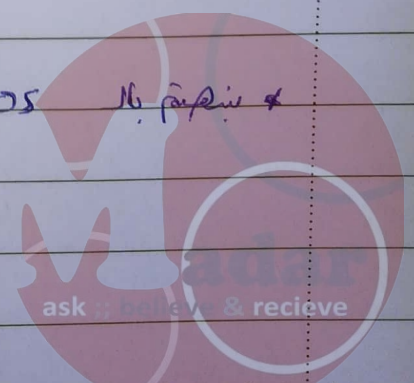
PP Kc is given, Kc is Range of stability



Controller parameter response



Response. No. of oscillations, OS, No. of peaks



* What is the value of Controller Gain ??

which Give $0.5 < 0.5$ (over shoot)

* Over shoot express about the Maximum deviation from Steady

* The criteria may be decay ratio or

Controller gain

* Criteria

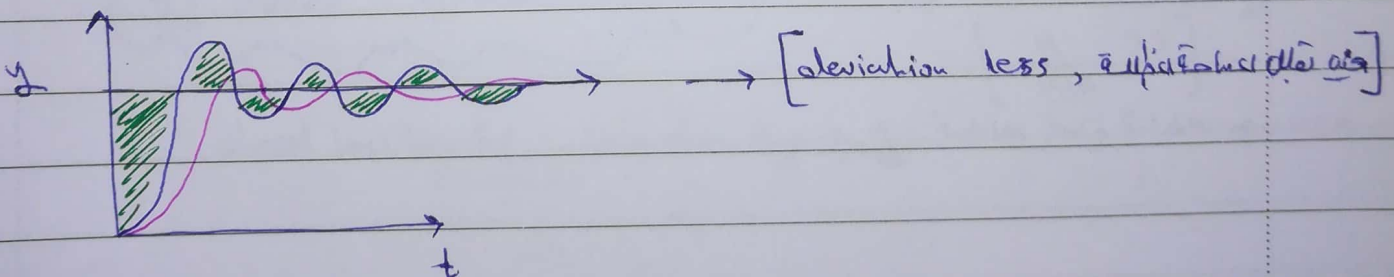
To minimize the over shoot and minimize the rise time.

$$OS = \exp \left[\frac{-2\pi}{\sqrt{1-\zeta^2}} \right]$$

→ No controller with these 2 condition. Because they are inversely

Comprehensive Criteria

Minimize the deviation as ~~overall~~ overall between set point and Response.



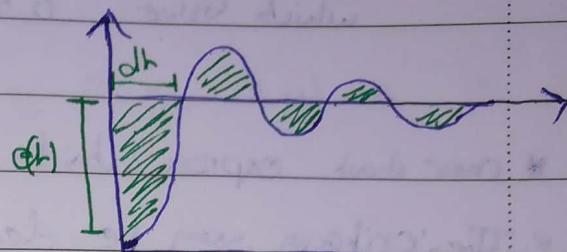
* Target of controller → To minimize the deviation between the set point and Response.

* * Minimize deviation → reduce the Area under the curve. *

I Integral Method →

To calculate the Area.

$$\text{Area} = \int_0^{\infty} e(t) dt$$

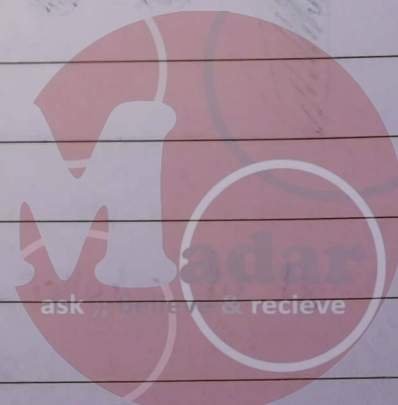


$$\text{ISE} = \int_0^{\infty} e^2(t) dt$$

up side -ve, +ve side
area is positive

→ If the error < 1 →

$$\text{IAE} = \int_0^{\infty} |e(t)| dt$$



* Comprehensive Design Criteria (less deviation or as all)

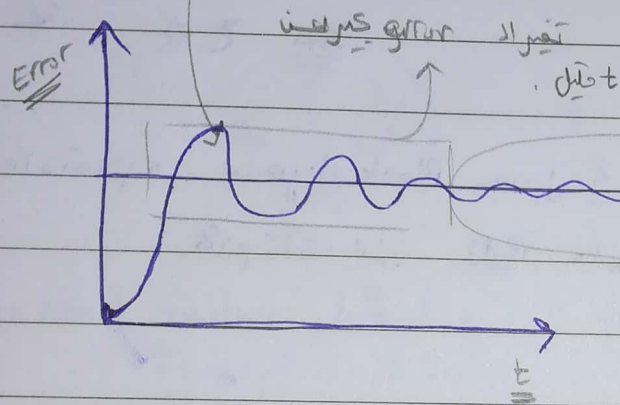
1] Integral Method

① $ISE = \int_0^{\infty} e^2 dt$

المساحة تحت المنحنى

② $IAE = \int_0^{\infty} |e| dt$

أثقل مساحة الخط كبيرة في البداية
الوزن الكبير



* كلما كان وقت الاستقرار أقل كلما كان أفضل
مع أقل التذبذب

③ $ITAE = \int_0^{\infty} t |e| dt$

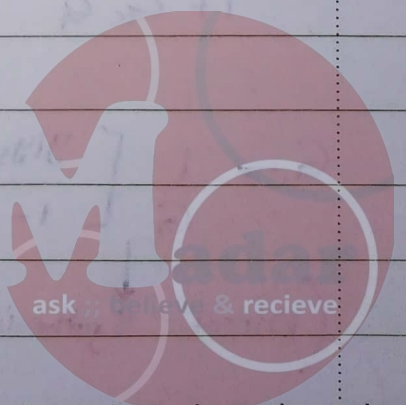
* استخدام طريقة ① ② ③ ، بالاعتماد على نوع controller

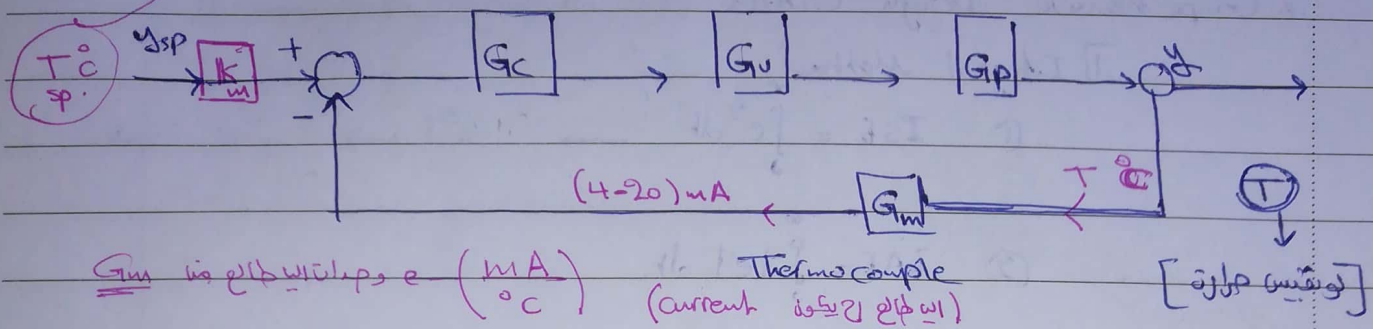
2] Direct Synthesis Method

كيف y هي اطلال بالمراد y_{sp}

$$\begin{bmatrix} y \\ y_{sp} \end{bmatrix}$$

closed loop transfer function. y مع y_{sp} هي y_{sp}

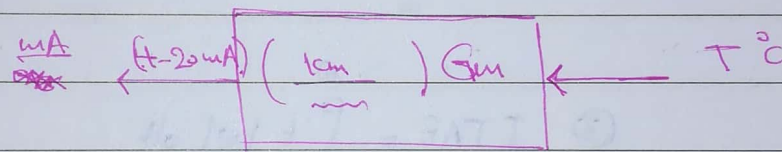




$$y = \frac{G_c G_v G_p}{1 + G_c G_v G_p G_m} y_{sp}$$

$$y_{sp} \approx y \approx \text{Error} *$$

Design is drawing, qualitative response Block diagram & transfer function
units & input to quantitative is B.D. & output



$$k_m = \frac{\Delta \text{output}}{\Delta \text{input}} = \frac{mA}{^{\circ}C}$$

* let $G_m = k_m$ and let $G = G_v G_p G_v$

Let G_c is the controller design for the system

$$y = \frac{G G_c}{1 + G G_c} y_{sp}$$

$$G_c = \frac{1}{G} \left[\frac{y_{sp}}{1 - y_{sp}} \right]$$

(Direct Synthesis Method)

(Performance criteria)

you have relationship between $G(s)$ of system and performance criteria and G_c

Spec G_c y_p y_{sp} (y/y_{sp})

↓
best performance

Direct

Transfer function with
zero order

* perfect control

$$\left[\frac{y}{y_{sp}} = 1 \right]$$

not achieve because of

$$G_c = \frac{1}{G} \left(\frac{1}{1-s} \right) \rightarrow$$



مستقر و بدون تاخیر

[استادان و دانشجویان]



Final??

* What another definition of perfect control ??

$$\left[\frac{y}{y_{sp}} = \frac{1}{s} = \infty \right]$$

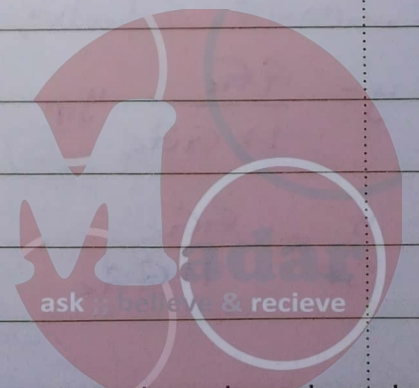
Zero order system process $\left[\frac{y}{y_{sp}} = 1 \right]$ is not

$$\left[\frac{y}{y_{sp}} = \frac{1}{Ts s + 1} \right]$$

(First order)

next step
(First order)

Zero order



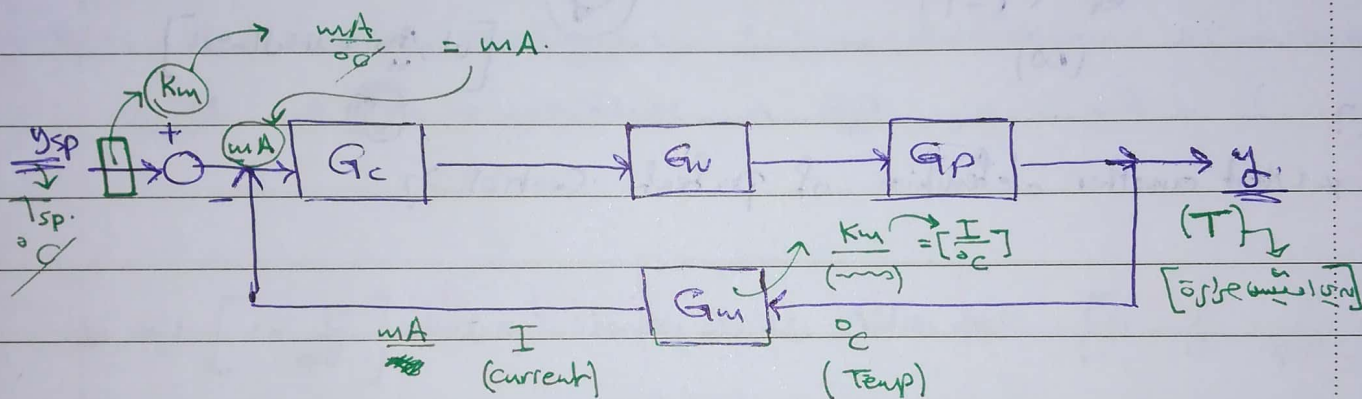
* Integral Method &

- ISE

- IAE

- ITAE

* Direct Synthesis Method



$$y = \frac{K_m G_c G_v G_p}{1 + G_c G_v G_p G_m} y_{sp}$$

(units & direction)

Unit conversion & quantitative relationship

* Let $G_m = K_m$

$$G_m G_p G_v = G$$

$$y = \frac{G G_c}{1 + G G_c} y_{sp}$$

Design of G_c for a given G

$$\frac{y}{y_{sp}} = \frac{G G_c}{1 + G G_c}$$

Solve the $G_c \Rightarrow G_c = \frac{1}{G} \left[\frac{y/y_{sp}}{1 - y/y_{sp}} \right]$

($G_m G_p G_v$)

ask, believe & receive

*** صفة كناية بي (مدح) فتارة، يجوز فيها العجالة فيقول ع

$\frac{y}{y_{sp}} = 1$ → perfect control! — means the dynamics between y and y_{sp}
@ zero order.

$$\frac{y}{y_{sp}} = \frac{1}{T_s + 1} \rightarrow \text{first order.}$$

$$G_c = \frac{1}{G} \left[\frac{\frac{1}{T_{cs}+1}}{1 - \frac{1}{T_{cs}+1}} \right]$$

$$G_c = \frac{1}{s} \cdot \frac{1}{T_{es}} \quad \#$$

$T_c \uparrow \rightarrow$ ideality is low

بجز T و T^m و T^m و T^m

(perfect) ideal π g.o.y

* الطريقة الثانية اختيار $PP = (y|y_{sp})$

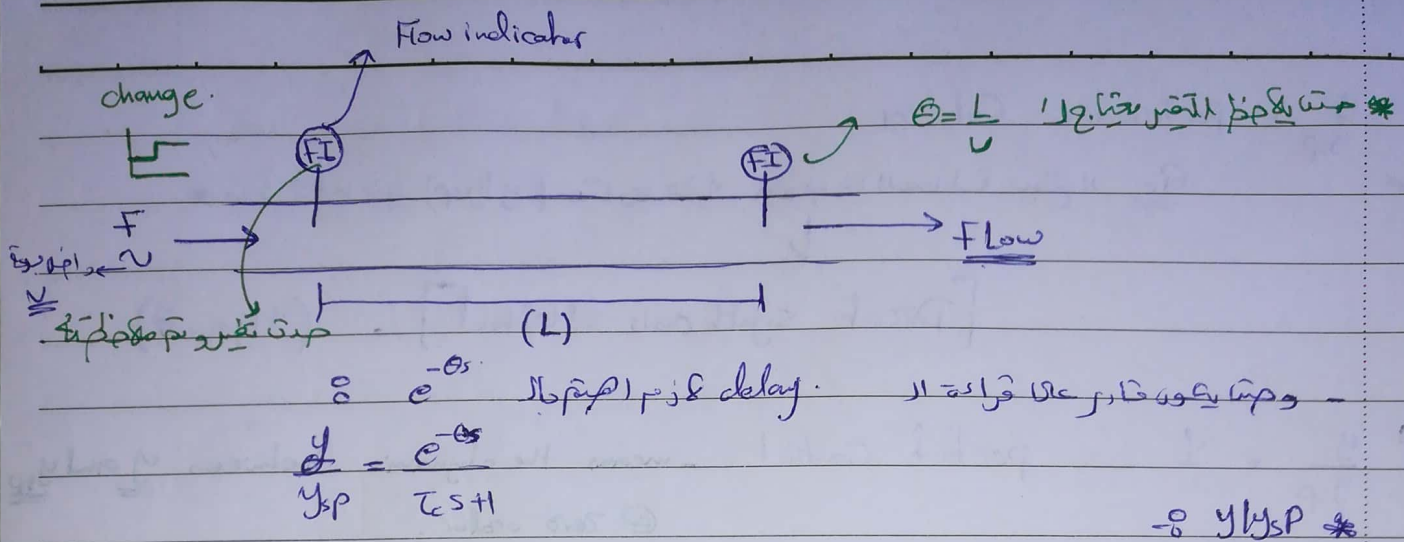
* How Can I choose the performance Gintaria (glysp) ??

→ (y_{lysp}) should include any characteristic important in process

میں، توپریاں ہیں ← delay
- میں design کے لئے

$\left(\frac{K_e = 0.5}{T_{s+1}} \right) \rightarrow \text{ajha up } [GIP] \text{ aur } 1 \text{ aur } 1 \text{ aur } 1$

$$\frac{M}{Y_{sp}} = \frac{1}{T_{est+1}} \rightarrow \text{delay in capture of first good QST order}$$



② Perfect

① First order

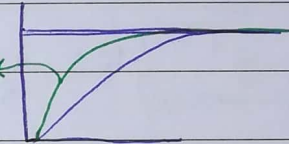
Process

② First order

③ Process

second order $\frac{k}{\tau^2 s^2 + \tau s + 1}$

How we can choose y_{sp} ?



$\frac{y}{y_{sp}} = \frac{1}{\tau s + 1}$

How we can choose y_{sp} ?

Physical delay. time

Ex 8: $G = \frac{2.3(1-0.5s)}{(s+1)(s+2)}$

design a [PI] controller for the system process using Direct Synthesis Method (DSM)

$G_c = \frac{1}{G} \left[\frac{y_{sp}}{1 - y_{sp}} \right]$

$\frac{y}{y_{sp}} = \frac{1}{\tau s + 1}$

$G_c = \frac{(s+1)(s+2)}{2.3(1-0.5s)} \cdot \frac{1}{\tau s}$

Possible roots → Control Unstable

This Controller Rejected Because it unstable

ask, believe & receive

* الاستيلاء يتم عبر احتلال هو y_{sp} حيث أعطاهم نسبة بمجموعة y في الدالة $unstable$

$$\Rightarrow \frac{y}{y_{sp}} = \frac{(1 - 0.55)}{1 + \tau_{cs}}$$
 - تم الغاء الدالة $stability$ في طريق الدالة $controllability$

$$\frac{y}{y_{sp}} = \frac{(1 - 0.5s)}{(1 + \tau_s)^2}$$

Importiers, Exportiers, Process || Import Trail anderer Länder

1. * Simple Criteria.
2. * Comprehensive Criteria.

Integral

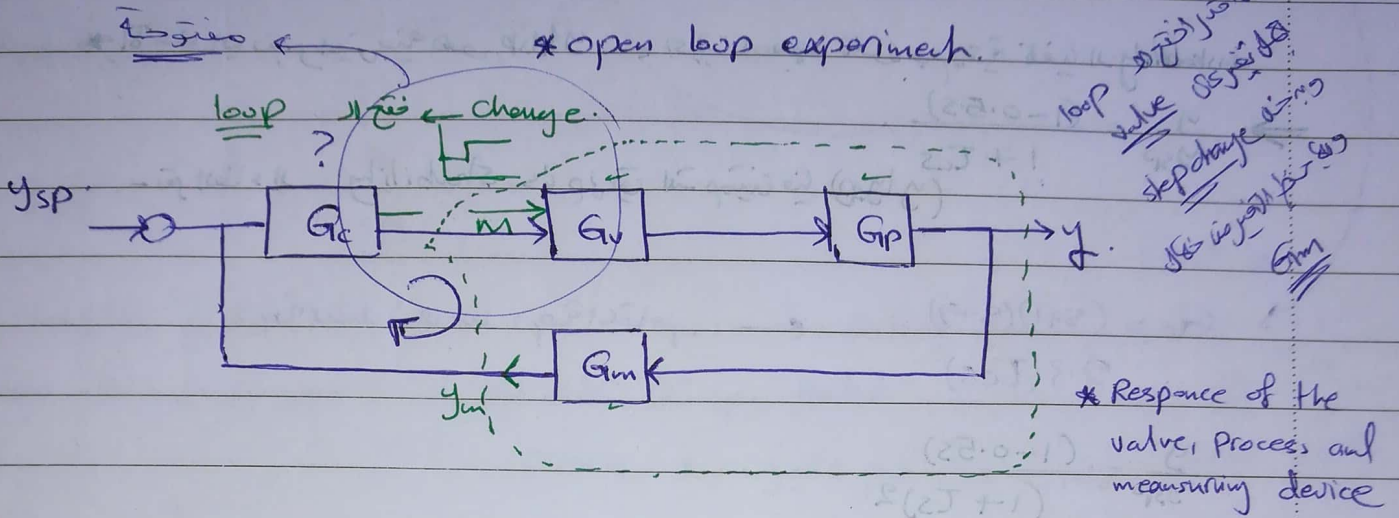
DSM

- 3] * Empirical Method \rightarrow From Experimental Test #
- Cohen - Coon \rightarrow open loop experimental
 - Ziegler - Nicols \rightarrow close loop experimental

Empirical Methods

I Cohen-Coon Method

* open loop experiment.



y_m output of the system is a step change in ①

(input) (output)

| t | M | y _m |
|---|---|----------------|
| 1 | 1 | 1 |
| 2 | 1 | 1 |
| 3 | 1 | 1 |

..., k, T, θ are data to be output. Fit ②

[Fit the data as 1st order + delay.]

$$G = G_v G_p G_m = \frac{K e^{-\theta s}}{T s + 1}$$

* Fitting data to the model
[k, θ, T]

project

* Fit for output and time

$$G_c = f(k, T, \theta)$$

controller is a function of
k, θ, T

| | |
|-----|---|
| P | $k_c = f_1(T, k, \theta)$ |
| PI | $k_c = f_2(k, T, \theta)$, $T_I = f_3(T, k, \theta)$ |
| PID | $k_c = f_4(k, T, \theta)$, $T_I = f_5(k, T, \theta)$, $T_D = f_6(T, \theta, k)$ |

Let the given system controller is known as T, k, θ are the parameters
 T_D, T_I, k_c are the parameters, k, θ, T are the process parameters

So $[T_I, T_D, k_c]$ are the parameters

① are the parameters

Final Simulation gives the response and initial value. up to the point ①
 now performance is the response of the system

2] Ziegler - Nichols Method

* close loop Test.

→ if y_{sp} is the set point change

, (k_c) proportional gain of the controller is known

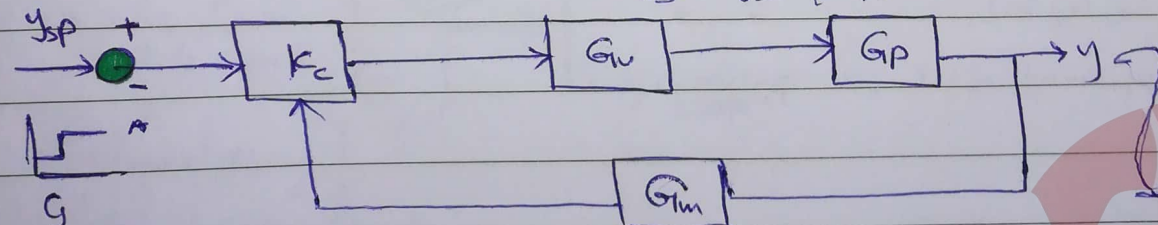
Let controller to be (P) controller

- use a small value of k_c (say $k_c = 1$)

- increase the value of k_c (say $k_c = 5, 10, \dots$)

← change the value

(the response of the system)



change on y_{sp}

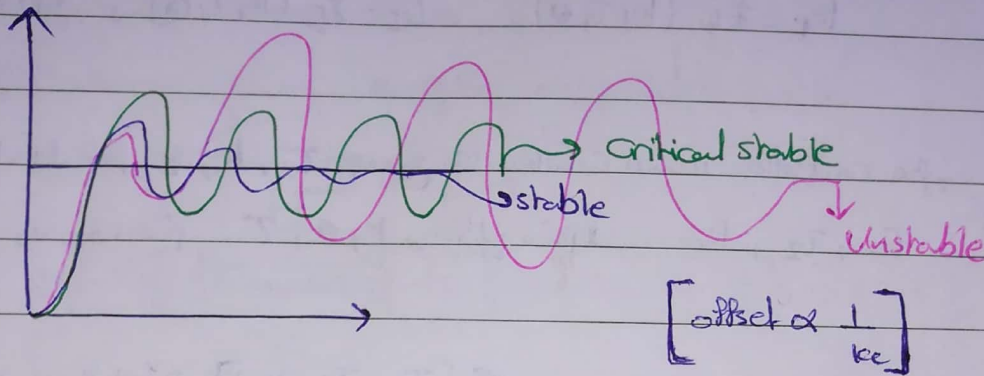
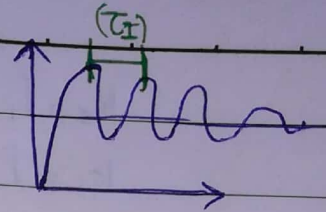
the output of the system

the response of the system

continuous oscillation

stable is system

✓ ادخال مقایسه stable لوز در قاعه طبع

[illegible]

π ← أقل حصة لـ
 لأنهم بدأوا لـ

$$k_c = \frac{k_{cu}}{2.2} \quad \text{مستطيل}$$

$T_I = \frac{P_n}{??}$ period of oscillation

Q. 10 Oscillation

Pin

$$k_c = \frac{k_m}{1.72}$$

$$I_T = \frac{P_{in}}{??}$$

$$T_D = \frac{K_A P_H}{??}$$

افلاو.

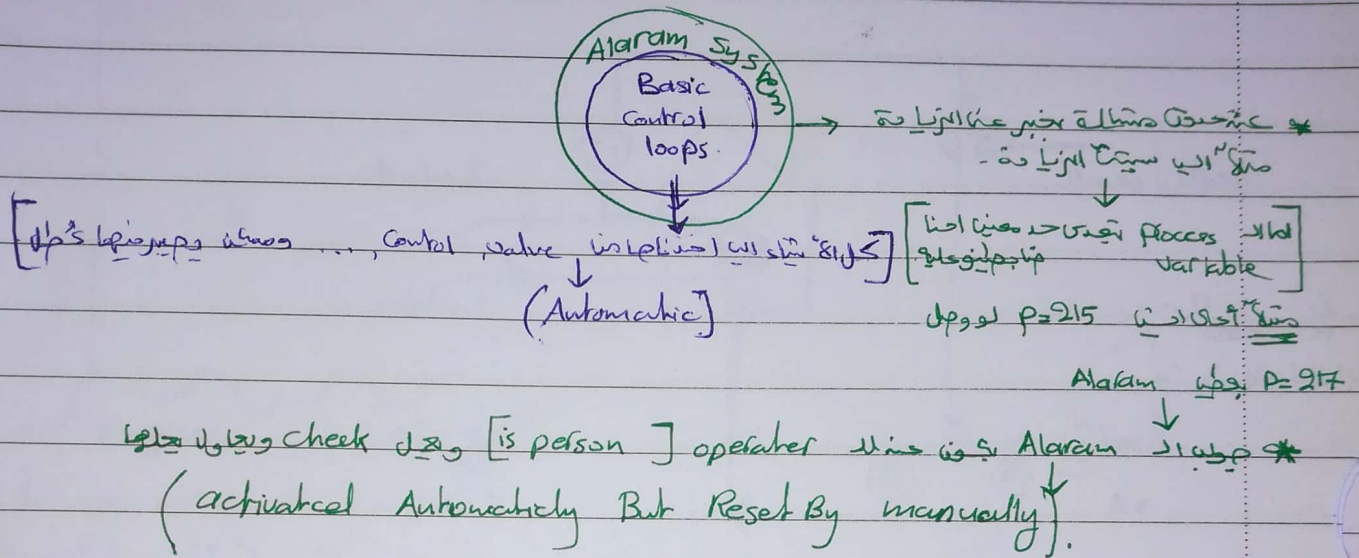
PI حقيقة كما أجبرنا على الـ

لأغراض طبية خاصة قاله ochillanin

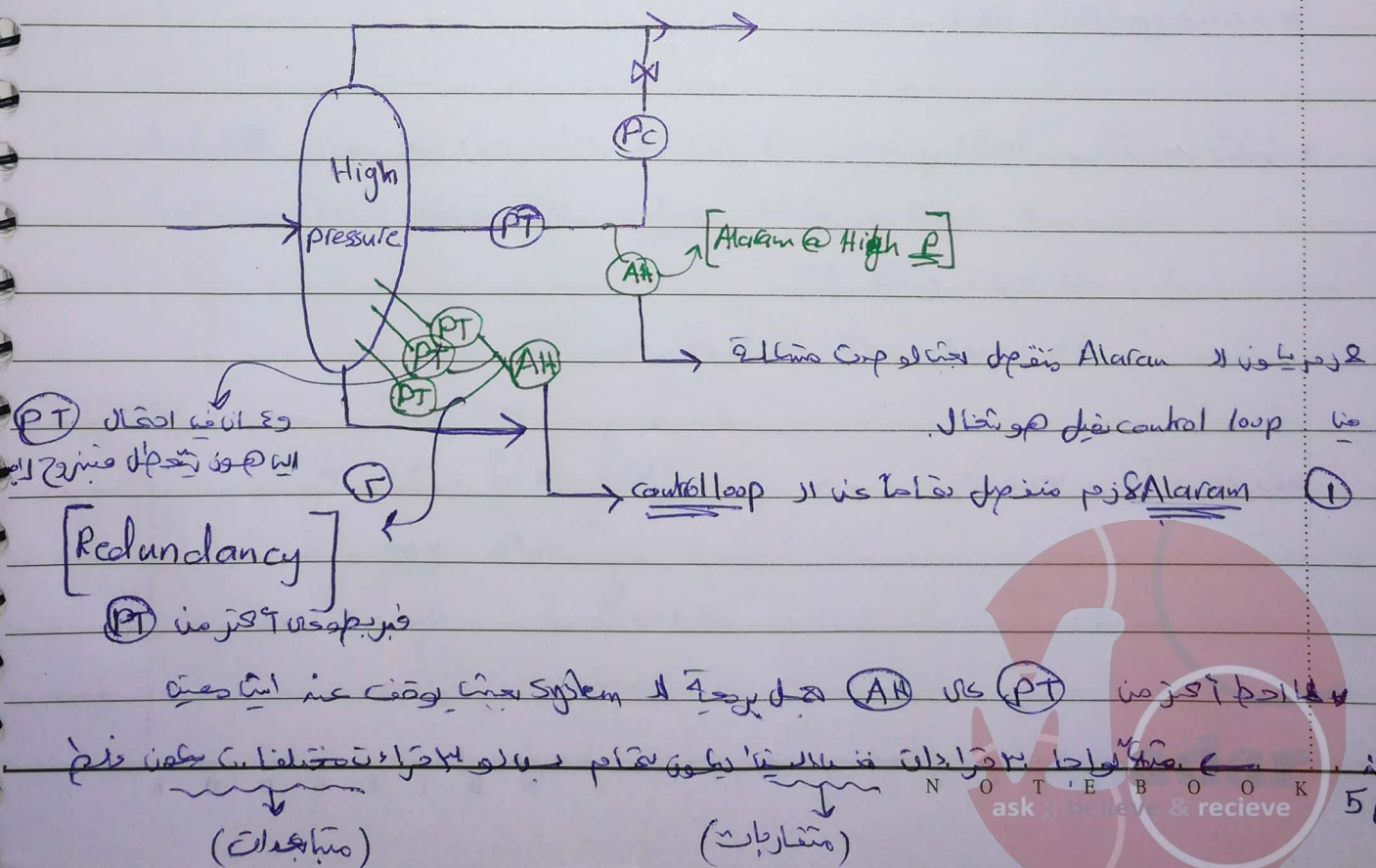
Application of Control :-

* Control system and Safety

① Redundancy



* Alarm System :-



Scanned by CamScanner

PP variable (P) variable *

Flow in line full is utility line [variable]

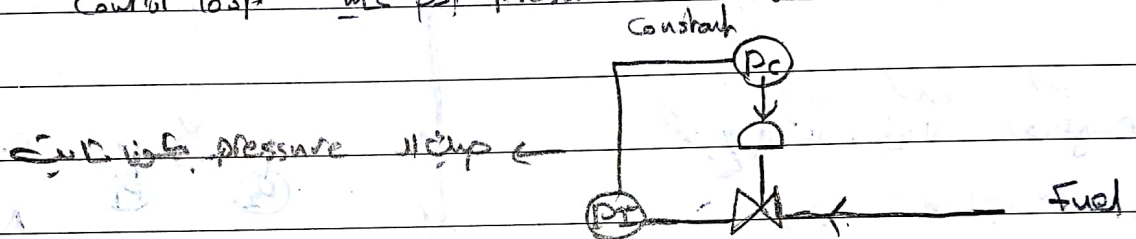
[pressure not constant]

There high probability variable pressure.

discharge * variable pressure

PP constant (DP) variable *

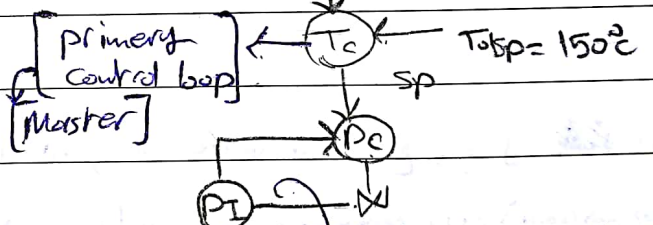
Control loop the pressure



PP variable pressure variable *

(T) temperature

Cascaded control
[2 control connect in series]



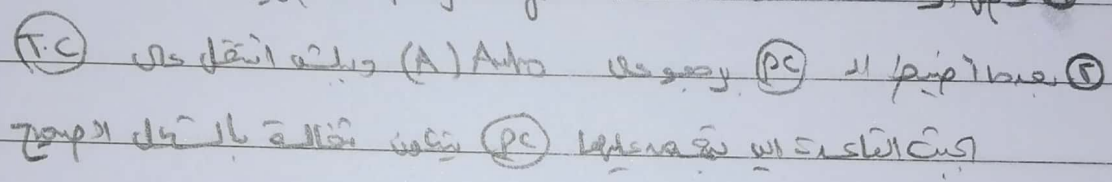
① manipulated variable
[Fluctuation] variable

main control loop variable

OR
primary control

[1 valve, 2 controller, 2 transducer] 2T single process

② dynamic is complicated and large

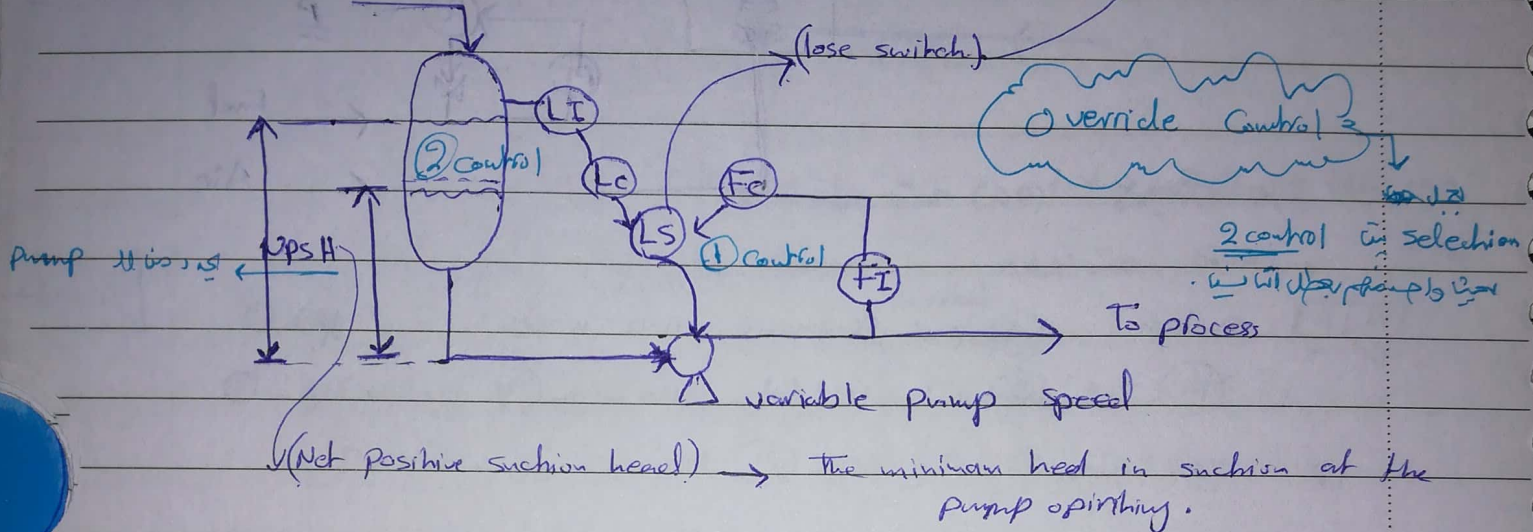


* selective Controllers

(give signal 116)

Several type of selective Control

- why selective
- selective of what



Positive suction head is required for flow of liquid into the pump

* ordinary control → open

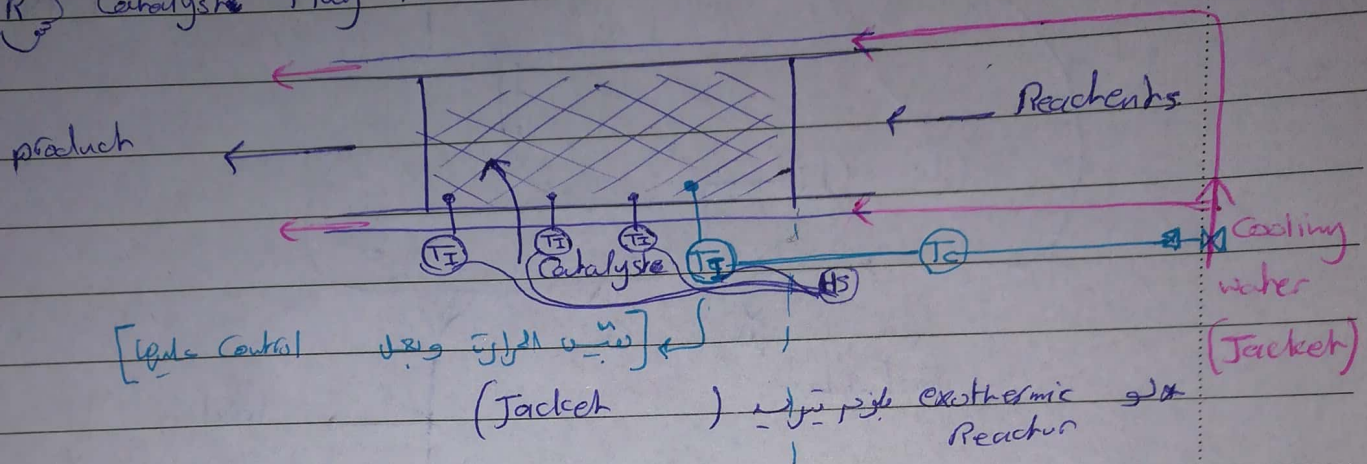
override control → during

Flow loop → NPSH → level → temperature → [override control]

(selective Between 2 controller)

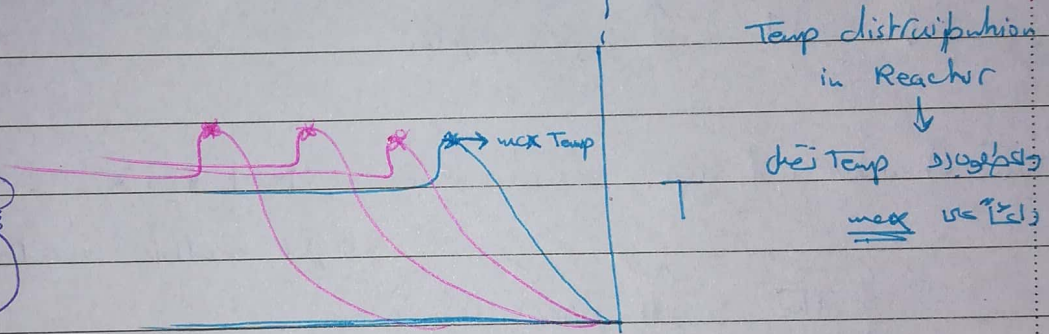
Selective of Control

PFR Catalyst Plug Flow Reactor.



* The objective of the control to maintain the Temp at the maximum possible Temp.

Selection of Transmitter



PP: 1. The first 50% of the catalyst bed is deactivated

deactivate position of max Temp function of the Catalyst activity.

position of max Temp function of the Catalyst activity.

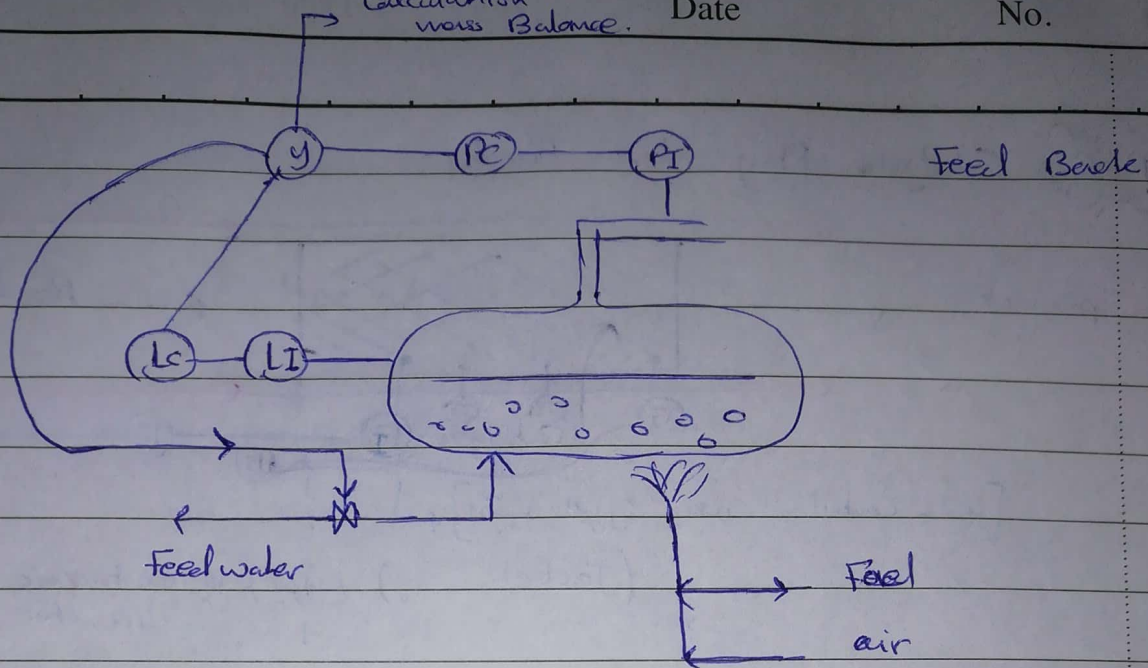
max activity not stationary

not stationary max Temp. (control)

* How can modify control loop?

(H5)) High selective is required Reactor

max



* if $P \downarrow$, $L \downarrow$ Because the evaporator increase.
 $P \downarrow \rightarrow L \uparrow$ \uparrow is just is just

* mix of water and vapor \rightarrow expansion \rightarrow level \uparrow just

vapor bubble exploded

The Swell phenomenon

just is just bubble pax is is just

* if $P \uparrow$, $L \downarrow$, vapor bubble collapsed

The shrink phenomenon

